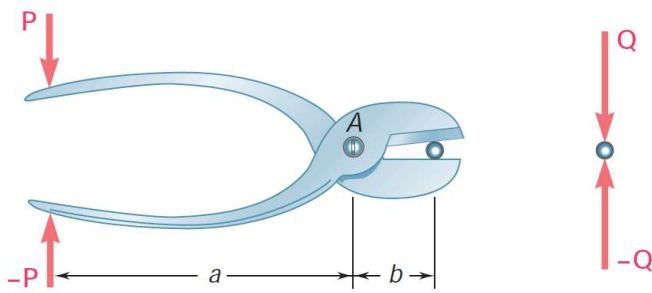


ANALYSIS OF MACHINES

Machines are structures designed to transmit and modify forces, whether they are simple tools or include complicated mechanisms.

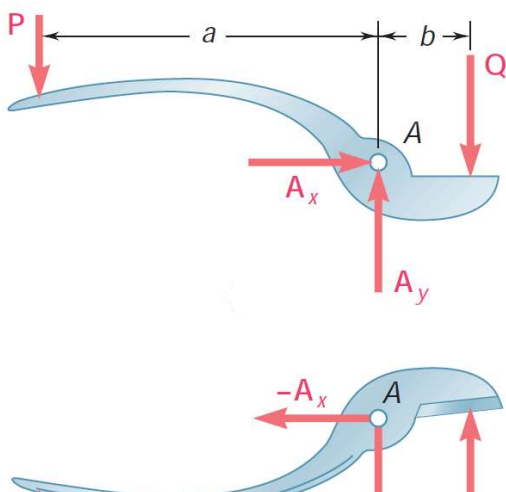
They simply transform input forces into output forces.



If we apply \vec{P} & $-\vec{P}$ on the handles of this cutting pliers, it will exert two equal & opposite forces \vec{Q} & $-\vec{Q}$ on the wire.

To determine Q for a given P (or conversely), we draw a free body diagram of the pliers. Note that the structure is nonrigid, hence we decompose it and analyse individual members.

Consider a FBD of one of the arms:-

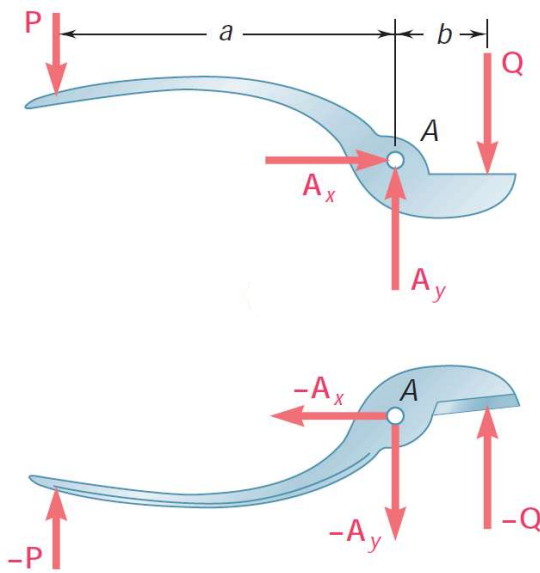


Taking moments about A:

$$P \cdot a = Q \cdot b$$

$$\sum f_x = 0 \Rightarrow A_x = 0$$

$$\sum f_y = 0 \Rightarrow A_y = P + Q$$



Taking moments about " "

$$P \cdot a = Q \cdot b$$

$$\sum F_x = 0 \Rightarrow A_x = 0$$

$$\sum F_y = 0 \Rightarrow A_y = P + Q$$

Example:

$$W = 1000 \text{ kg}$$

The hydraulic lift consists of two identical linkages on which hydraulic cylinders exert forces.

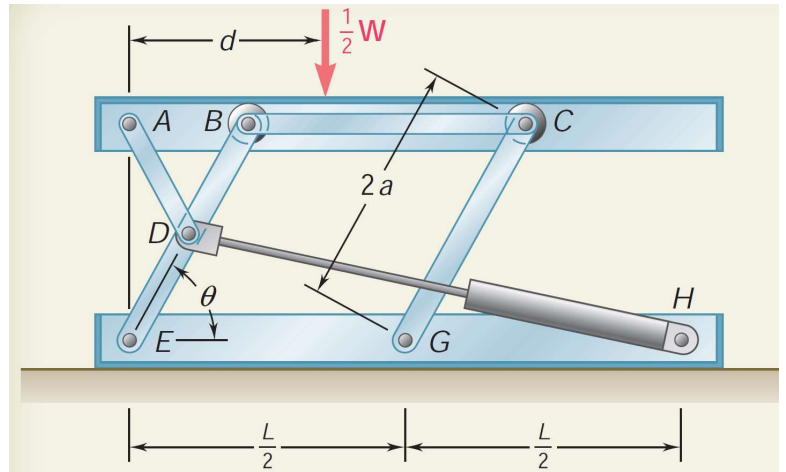
One linkage & cylinder is shown.

Members EDB & CH are of length $2a$ each.

AD is pinned at midpoint of EDB.

Determine force exerted by the cylinder in raising the crate for $\theta = 60^\circ$, $a = 0.7 \text{ m}$, $L = 3.20 \text{ m}$.

Is the result independent of d ?

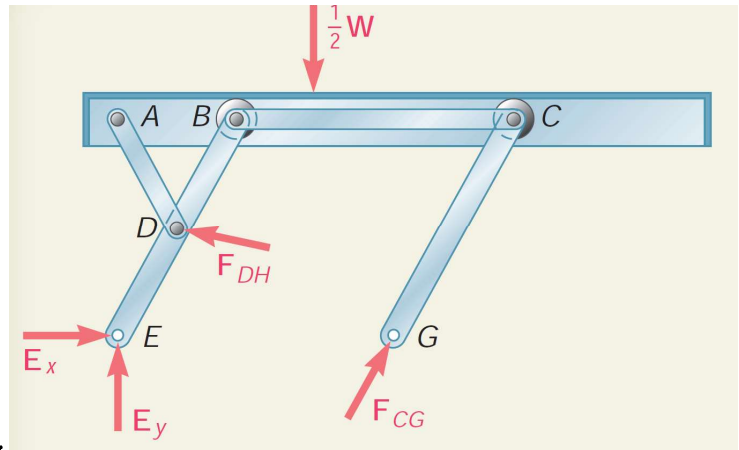


Sol: FBD of the machine!

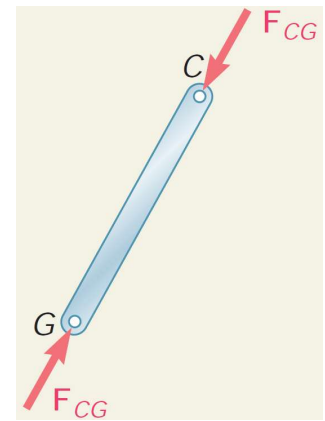
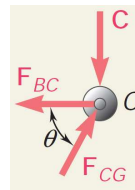
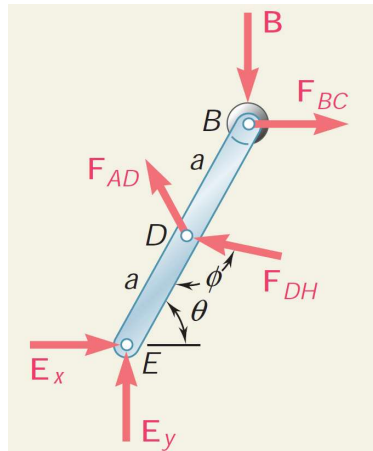
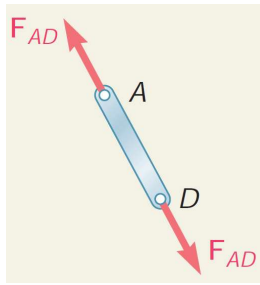
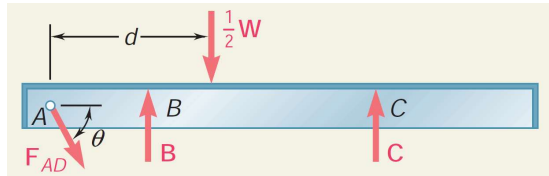


~~1.5~~
machine!

There are more than 3 unknowns, hence we need to dismember the machine.



AD, BC, CG: Two force members
 CG: Assumed to be in compression.

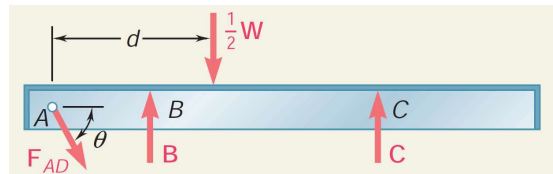


Platform ABC:

$$\sum F_x = 0$$

$$\Rightarrow F_{AD} \cos \alpha = 0 \Rightarrow F_{AD} = 0$$

$$\sum F_y = 0 \Rightarrow B + C = \frac{1}{2} W$$



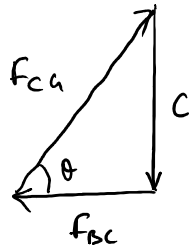
Roller C:

1 ... triangle!



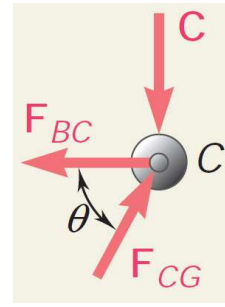
Koller C.

We can draw a force triangle:



$$\tan \theta = \frac{C}{f_{bc}} \Rightarrow$$

$$f_{bc} = C \cot \theta$$



Member BDE:

$$\sum M_E = 0$$

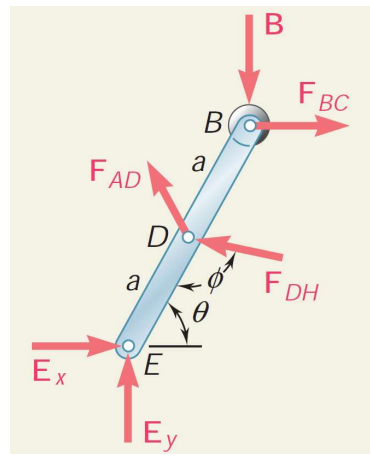
$$\Rightarrow F_{DH} \cos(\phi - 90^\circ) a - B(2a \cos \theta) - f_{bc}(2a \sin \theta) = 0$$

$$\Rightarrow F_{DH} \sin \phi - 2(B+C) \cos \theta = 0$$

Using $B+C = \frac{1}{2}W$, we get

$$F_{DH} = W \frac{\cos \theta}{\sin \phi}$$

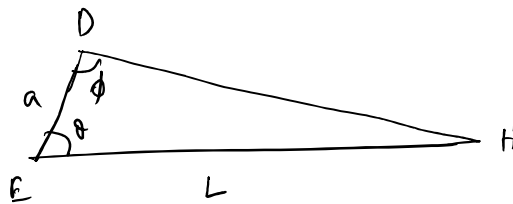
F_{DH} is independent of d .



Using law of cosines: $(DH)^2 = a^2 + L^2 - 2aL \cos \theta$

$$\Rightarrow DH = 2.91 \text{ m}$$

Law of sines: $\frac{\sin \phi}{EH} = \frac{\sin \theta}{DH} \Rightarrow \sin \phi = \frac{EH}{DH} \sin \theta$



$$f_{DH} =$$