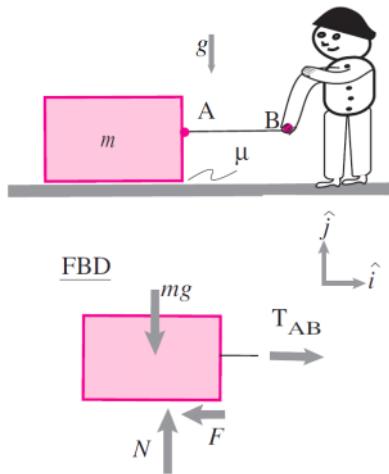


EQUILIBRIUM WITH FRICTIONAL CONTACT

We will only deal with Coulomb friction with a single coefficient of friction, μ , i.e., $M_{\text{static}} = M_{\text{dynamic}}$ is the approximation we will use.



Dragging a block with friction:-

From force balance, from FBD, given:

$$\sum \vec{f}_i = \vec{0} \Rightarrow -mg \hat{j} + T_{AB} \hat{i} + N \hat{j} - f \hat{i} = \vec{0} \quad (a)$$

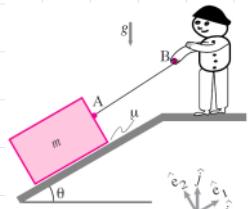
$$\text{We will use } F = \mu N \quad (b)$$

Equations (a) & (b) are three equations for three unknowns: F , N & T_{AB} .

NOTE: for particle mechanics, we don't need to worry where \vec{N} & \vec{F} are applied.

Dragging a block on a ramp with friction:-

- We can either
- Slide the block 'up' with rod AB
 - Push the block 'down' with rod AB
 - Hold it still.



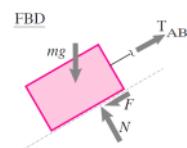
From force balance:-

$$\sum \vec{f}_i = \vec{0}$$

$$\Rightarrow -mg \hat{j} + T_{AB} \hat{i}_1 + N \hat{i}_2 - f \hat{i}_1 = \vec{0} \quad (c)$$

2 scalar equations.

Equation (c) along with friction relation gives us 3 equations for 3 unknowns in T_{AB} , F and N .



$$\begin{aligned} \{\text{Eqn } ①\} \cdot \hat{e}_1 = 0 &\Rightarrow -mg \sin \theta + T_{AB} - F = 0 \\ \{\text{Eqn } ①\} \cdot \hat{e}_2 = 0 &\Rightarrow mg \cos \theta + N = 0 \end{aligned}$$

Standard Coulomb friction model :-

- (i) $F = \mu N$ if the block is sliding up
- (ii) $F = -\mu N$ if the block is sliding down
- (iii) $-\mu N \leq F \leq \mu N$ if the block is not sliding

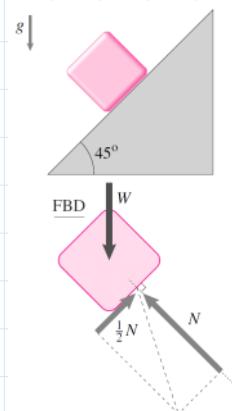
Solving eqn. ① with the above frictional relations given :

- (i) $T_{AB} = mg(\mu \cos \theta + \sin \theta)$ if block is sliding up
- (ii) $T_{AB} = mg(-\mu \cos \theta + \sin \theta)$ if block is sliding down
- (iii) Note that if $\tan \theta < \mu$, then $T_{AB} < 0 \Rightarrow$ you'll have to push the block to slide it down.
- (iii) $mg(-\mu \cos \theta + \sin \theta) \leq T_{AB} \leq mg(\mu \cos \theta + \sin \theta)$ if the block is not sliding.

If $\tan \theta < 0$, then $T_{AB} = 0$ is a solution for no sliding, i.e., the block sits still on the ramp without pulling on the rod.

No forces solution :

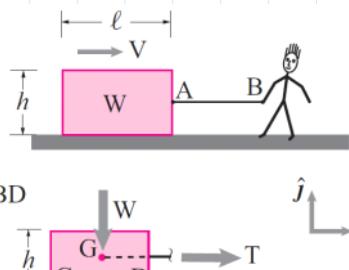
A block on a ramp with angle 45° & with $\mu = 0.5$ cannot be in static equilibrium !



Block as an extended body :-

Does dragging force on the block?

~ FBD.

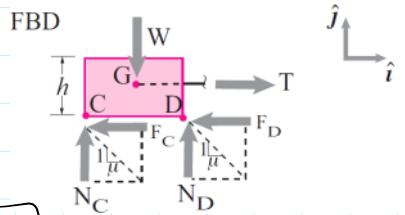


block?

From FBD:

$$N_C + N_D = W$$

$$\text{and } \underbrace{F_C}_{\mu N_C} + \underbrace{F_D}_{\mu N_D} = T_{AB} \Rightarrow \boxed{T_{AB} = \mu W}$$



Taking moment about C gives:

$$N_D = \frac{W}{2} + \frac{\mu h W}{2L}$$

$$N_D = \frac{W}{2} - \frac{\mu h W}{2L}$$

\Rightarrow There is more pressure at D than at C.

Condition of Contact, Consistency & Contradiction!:-

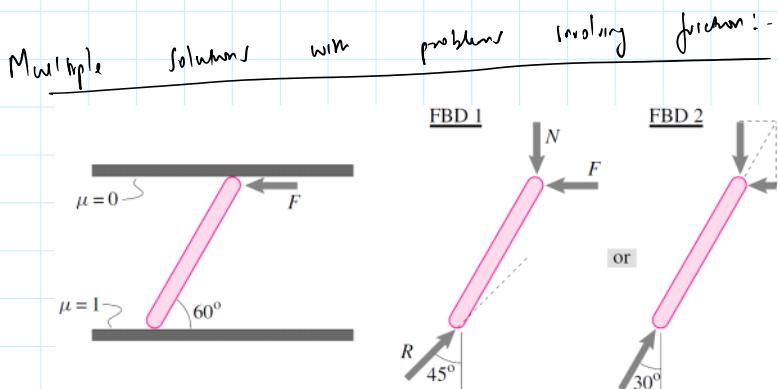
Problems involving 'Contact' tend to be conditional in nature:-

- Ex: (i) Ground pushes up on the object to prevent interpenetration. If the pushing is positive, else the ground does not push up the body.
- (ii) force of friction $= \mu N$ if there is slip,
else the force is $< \mu N$.
- (iii) Distance between two points is kept down increasing if tension in the string is positive, else tension is zero.

Rules of thumb:-

On a FBD at every point of frictional contact

- If the direction of slip or impending slip is known, either
 - Draw a normal force N and a friction force $F = \mu N$ opposing the relative slip, or
 - Draw a single force R at an angle ϕ from the normal of the contact in the direction which resists slip (with $\tan \phi = \mu$)
- If there is no slip, either
 - Draw a normal force N and tangential force F or
 - Draw a single force vector \vec{R} with unknown components
- If you don't know whether or not there is slip, first
 - Guess that there is no slip, then
 - Solve the equilibrium equations, then
 - * If $F \leq \mu N$: you guessed right and have found a solution to both the equilibrium and friction equations.
 - * If $F > \mu N$: you guessed wrong and have to guess that there is slip in one direction (guess which), then
 - see if you can solve the equilibrium equations, if not then
 - assume slip in the opposite direction and try to solve the equilibrium equations, if you can't, then
 - the problem has no solution



Consider a weightless rod which is just long enough to make 60° angle with the walls of a channel. Lower wall has $\mu = 1$ and upper wall is frictionless. What is the force needed to keep it in equilibrium?

If the rod is assumed to be sliding, we get FBD 1. The form shown can be in equilibrium if all the forces are zero. So a solution is that the rod slides in equilibrium with no force.

If the rod is not sliding, the friction force on the lower wall can be at any arbitrary angle since we require $|F_x| \leq \mu N_x = N_x \Rightarrow -N_x \leq F_x \leq N_x$ where N_x & F_x are reaction and frictional forces on the lower wall.

$$N_u : \text{Reaction at upper wall.} \\ \text{for equilibrium, we have } N_x = N_u \\ \therefore F = F_x.$$

∴ Reaction on the lower wall can

$$4 \quad F = f_x$$



We can equilibrium in FBD 2

The friction force on the lower wall can be at any angle between $\pm 45^\circ$.
for arbitrary positive F .

Summary: We can either have fully slipping solutions with no force or jammed (stuck) solutions with arbitrary force.

This physically corresponds to one being able to easily slide a rod like this down a slot and then also at a different instant, have the rod totally jam. Such "soft-locking" rods are used in some rock-climbing equipment.

Statistically indeterminant problems:-

When two or more points of contact have friction, then statistical indeterminacy is likely if there is no slip.

Ex: Person sitting in Chair

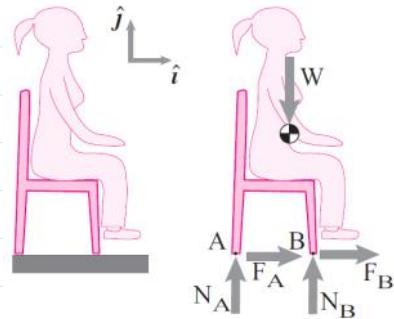
$$|F_A| \leq \mu N_A$$

$$\text{and } |F_B| \leq \mu N_B$$

Equilibrium:

$$f_A + f_B = 0$$

$$\text{and } N_A = N_B = \frac{W}{2} \quad (\text{Assume } W \text{ is in the middle})$$



All we can say is

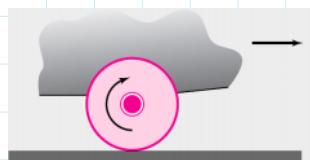
$$-\frac{W}{2} \leq f_A \leq \frac{W}{2} \quad \text{and } f_B = -f_A$$

Wheel as a two-force body:

Without a wheel, it takes a force of approximately ' MW ' to drag something weight W .

For trolley, $M \approx 0.1$ & for

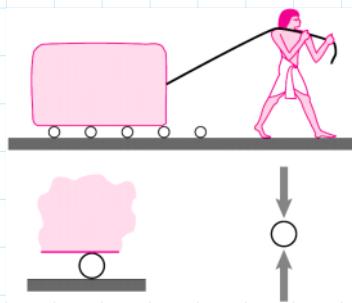
stone on ground, $M \approx 0.6$ and, $M \approx 1$ for stone on ground.



One way to reduce friction is to have rolling logs below the cart.



This idea has a deficiency - logs have



free-body diagram of the wheel:-

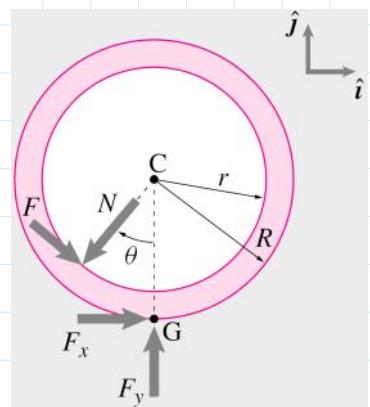
Assumption: Wheel weight is assumed negligible.

r : Radius of the wheel (shown here to be very large)

R : Radius of the wheel.

This idea has a deficiency - logs have to move from back to the front of the boulder.

One way to avoid this is to introduce a fixed non-rotating shaft to a circular disk with a hole - a wheel.



The force of the axle on the wheel has a normal component N and a frictional component F . The forces due to the ground are F_x and F_y .

Unknowns: N, F, θ, F_x .

F_y is determined by the weight of the cart.

$$\underline{\text{Equations:}} \quad F = \mu N \quad (\text{frictional relation})$$

$$\underline{\text{Force balance:}} \quad F_x \hat{i} + F_y \hat{j} + N (-\sin \theta \hat{i} - \cos \theta \hat{j}) + F (\cos \theta \hat{i} - \sin \theta \hat{j}) = 0$$

$$\underline{\text{Moment about C:}} \quad FxR + f_x R = 0$$

We are mainly interested in finding the resultant force $\underline{F_x}$ in the ground.

We can solve the four scalar equations for the four unknowns F_x, N, F , and θ in terms of μ, R, F_y and μ .

$$\underline{\text{Equations:}} \quad F = \mu N \quad \dots \quad (1)$$

$$F_x - N \sin \theta + F \cos \theta = 0 \quad \dots \quad (2)$$

$$F_y - N \cos \theta - F \sin \theta = 0 \quad \dots \quad (3)$$

$$F_x + f_x R = 0 \quad \dots \quad (4)$$

$$\text{From } (1), (2) : \quad f_x - N \sin \theta + MN \cos \theta = 0 \quad \dots \quad (5)$$

$$\text{From } (1) + (3) : \quad F_y - N \cos \theta - \mu N \sin \theta = 0 \quad \dots \quad (6)$$

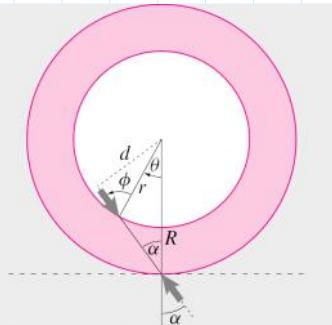
$$\text{From } (6) : \quad F_y = N (\cos \theta + \mu \sin \theta) \quad \dots \quad (7)$$

$$(2) \times \cos \theta - (3) \times \sin \theta : \quad f_x \cos \theta - F_y \sin \theta - \cancel{N \sin \theta \cos \theta} + \cancel{N \cos \theta \sin \theta} \\ + F \cos \theta \cdot \cos \theta + F \sin \theta \cdot \sin \theta = 0$$

$$\Rightarrow F_x \cos\theta - F_y \sin\theta + F = 0 \quad \text{--- (8)}$$

We can follow a slightly different approach to obtain M_{yy} !

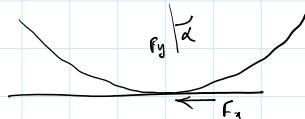
The wheel is really a two-force body. So for equilibrium, we require two equal & opposite collinear forces at the two contact points as shown.



ϕ : friction angle, i.e., $\mu = \tan \phi$
where μ is coeff. of friction between axle and wheel.

α can be thought of as a friction angle due to an "effective" friction at the wheel.

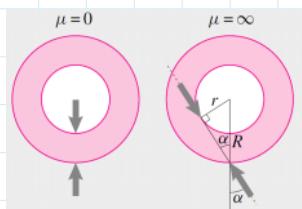
Note that α is not the sliding friction coeff. between wheel & ground, since we assume that the wheel is in pure rolling motion.



Instead, $M_{yy} = \tan \alpha$ is the specific resistance or the coefficient of rolling friction.

If there was no wheel, $M_{yy} = \mu$.

Extreme limits: $M = 0$ and $M = \infty$.



$M = 0$: frictionless bearing $\Rightarrow \phi = 0^\circ$

No ground resistance.

$M = \infty$: Infinite friction coeff $\Rightarrow \phi = 90^\circ$.

Here we get $\sin \alpha = \frac{g}{R}$

Thus if an axle has a diameter of 10 cm & a wheel of 1m,
then $\sin \alpha = 0.1/1$. For such small values of $\sin \alpha$,
we can take $\tan \alpha \approx \sin \alpha \approx M_{yy}$.

From the two-force body FBD, we have

$$\begin{aligned} \underbrace{\sin \phi}_{d} &= \underbrace{R \sin \alpha}_{d} \\ \Rightarrow \sin \alpha &= \frac{g}{R} \end{aligned}$$

Limiting cases:

$$\phi = 0 \Rightarrow \sin \phi = 0 \Rightarrow \tan \alpha = 0 \Rightarrow M_{xy} = 0$$

$$\phi = 90^\circ \Rightarrow \sin \phi = 1 \Rightarrow \tan \alpha = \frac{R}{\lambda}$$

$$\Rightarrow \tan \alpha = \frac{\sin \phi}{\sqrt{1 - \sin^2 \phi}} = \frac{R}{\lambda} \frac{1}{\sqrt{1 - \frac{\lambda^2}{R^2}}} = \frac{R}{\sqrt{R^2 - \lambda^2}}$$

If $\lambda \ll R$, & μ is small, $\sin \phi \approx \tan \phi = \mu$

$$\therefore \tan \alpha \approx \mu \Rightarrow M_{xy} = \frac{\lambda}{R} \cdot \mu$$

We can now combine the wheel with the rolling logs "inside" to get a ball-bearing wheel.



Each ball is a two-force body and thus only transmits radial loads.

Real ball bearings are not perfectly smooth nor perfectly rigid, so to obtain small M_{xy} , we should ideally keep $\frac{\lambda}{R} \ll 1$.

Example: Ramp on block sliding up or down:-

$$m = 10 \text{ kg}$$

Block pushed up by force F .

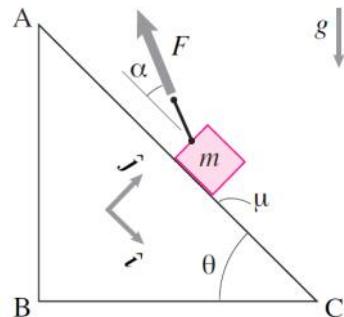
$$\mu = 0.7$$

① Let $\theta = 60^\circ$ & $\alpha = 0^\circ$. If block slides down, what is the tension in the string.

② Let $\theta = 30^\circ$ & $\alpha = 30^\circ$. If $F = 20 \text{ N}$, what is the friction on the block?

③ Let $\theta = 60^\circ$ & $\alpha = 30^\circ$. If $F = 10 \text{ N}$, what is the friction on the block?

④ for $\theta = 30^\circ$ & $\alpha = 30^\circ$, what will be the tension in the string to make the block just slide up on the slope? Express your answer in terms of the weight of the block.



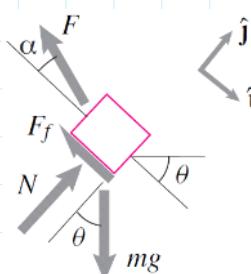
Solution: $\sum \vec{F} = \vec{0}$

$$\Rightarrow \vec{F} + \vec{N} + \vec{F}_f + \vec{W} = \vec{0}$$

$$\Rightarrow F(-\cos \alpha \hat{i} + \sin \alpha \hat{j}) + N \hat{i} + mg(\sin \theta \hat{i} - \cos \theta \hat{j}) = \vec{0}$$

①

① Block sliding down:- If sliding rate is steady & very slow, then $\vec{a} \approx 0$. Also, $\vec{F}_f = -\mu N \hat{i}$



From ①, we have:

$$-F \cos \alpha - \mu N + mg \sin \theta = 0$$

$$F \sin \alpha + N - mg \cos \theta = 0$$

Eliminating N , we have

$$F = \left(\frac{mg \cos \theta - \mu g \sin \theta}{\cos \alpha - \mu \sin \alpha} \right) mg$$

Substituting the numerical values, we get $\boxed{\vec{F} = (-51N)\hat{i}}$

② Block sliding or not sliding - not known:-

$$F = 20N \quad \alpha = 30^\circ, \theta = 30^\circ$$

Since we do not know if the block is sliding or not.

Let us assume static equilibrium in the given configuration

& solve for the friction force F_f . We will then check

if it satisfies friction law for static equilibrium ($|F_f| \leq \mu N$).

Substituting $\vec{F}_f = -F_f \hat{i}$, we get

$$\begin{aligned} -F \cos \alpha - F_f + mg \sin \theta &= 0 \\ F \sin \alpha + N - mg \cos \theta &= 0 \end{aligned} \quad \left. \begin{array}{l} F, \alpha, \theta, mg \\ \text{are known.} \end{array} \right.$$

$$F_f = mg \sin \theta - F \cos \alpha$$

$$\text{&} N = mg \cos \theta - F \sin \alpha$$

Putting in the numbers, $F_f = 31.73N$
& $N = 74.96N$.

$$\begin{aligned} \text{Maximum possible value of friction force is } \mu N &= 0.7 \times 74.96N \\ &= 52.47N. \end{aligned}$$

Thus $|F_f| < \mu N$, and therefore, our assumption of static equilibrium is valid.

$$\text{Thus } \vec{F} = -31.73N \hat{i}$$

③ Block sliding or not sliding - not known, again!:-

$$F = 10N, \alpha = 30^\circ \quad \text{&} \theta = 60^\circ$$

Again, assuming static equilibrium, we get

$$F_f = 76.3N \quad \text{&} N = 44N$$

But $|F_f| > \mu N$. Clearly, F_f is not less than μN equal to μN , & therefore our assumption of static equilibrium is not valid. In fact, for the given parameters, the block will accelerate downhill - a problem of dynamics.

However, the friction force remains constant, at its maximum value of $\vec{F}_f = \mu N = 33.88N$ till the sliding starts, accelerating or not.

$$\vec{F}_f = -33.88N \hat{i}$$

④ Block just about to slide upward!:-

$$\text{Now } F_f = \mu N.$$

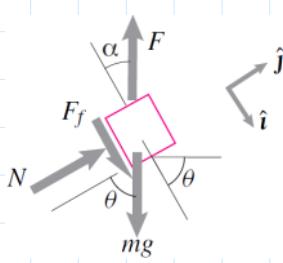
$$-F \cos \alpha + \mu N + mg \sin \theta = 0$$

$$F \sin \alpha + N - mg \cos \theta = 0$$

Eliminating N , we get

$$F = \left(\frac{\sin \theta + \mu \cos \theta}{\cos \alpha + \mu \sin \alpha} \right) mg$$

$$= \left(\frac{1.216}{1.216} \right) mg = mg.$$



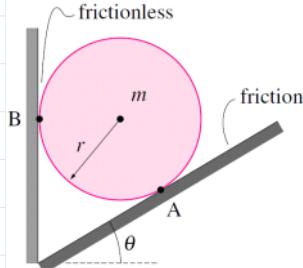
Does the answer makes sense? Yes, it does.

For given θ & α , the string tension is vertical. If it balances the weight of the block, normal force goes to zero & so does the friction force.

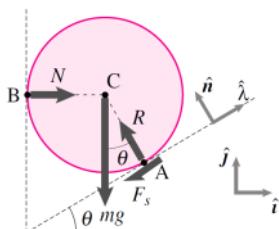
If tension increases by a tiny amount, the block tends to slide up.

Example: How much friction does the cylinder need?

Find the friction force on the cylinder from the incline.



Free-Body diagram:-



We need to find F_s .

$$\sum \vec{M}_c = 0 \Rightarrow \tau F_s (-R) = 0 \\ \Rightarrow F_s = 0$$

Force of friction on the cylinder is zero.

In fact F_s is independent of θ .

Note: The cylinder is a 3-force body (two contact forces at A & B and one gravity force).

