

Vehicle Dynamics Term Project Driveline Modelling

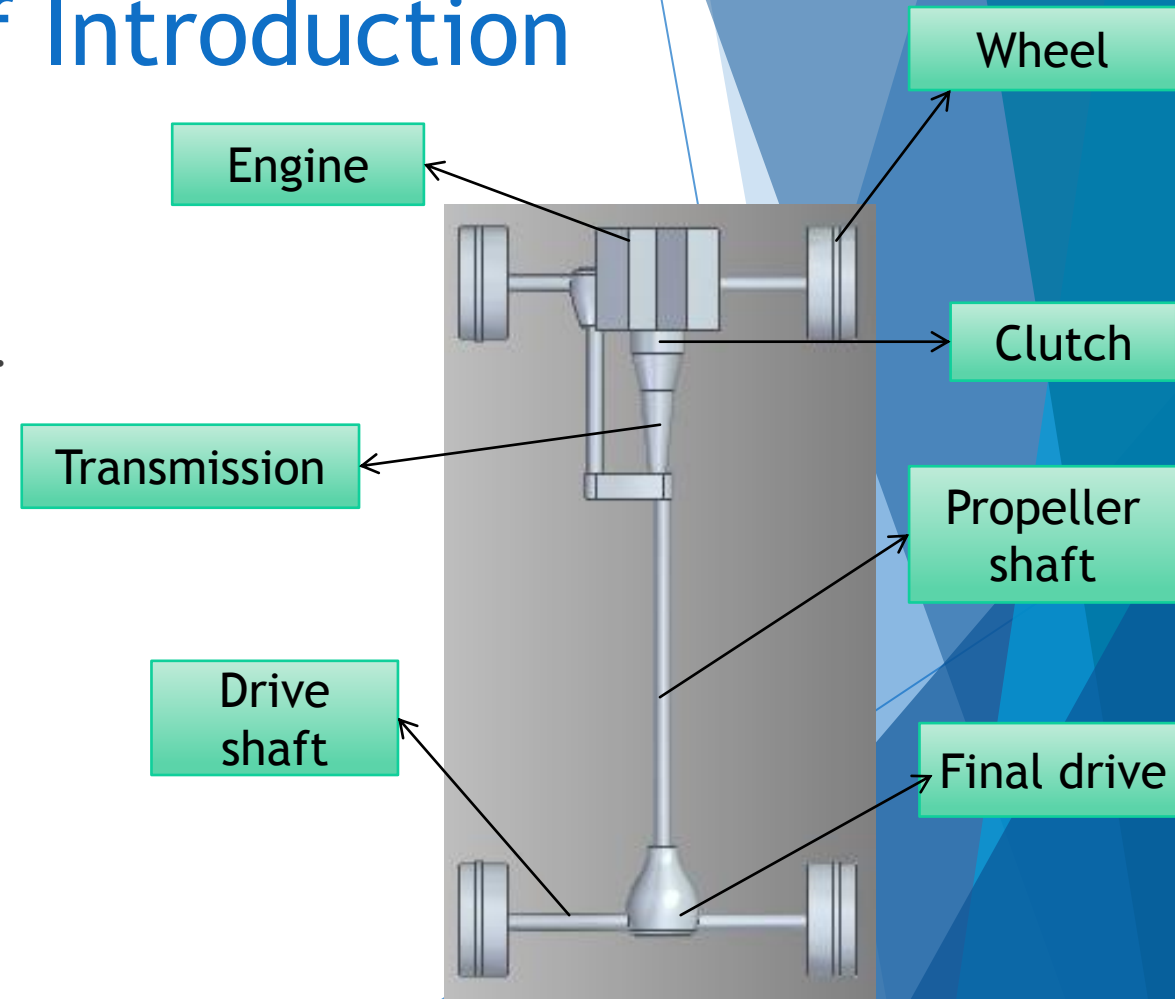
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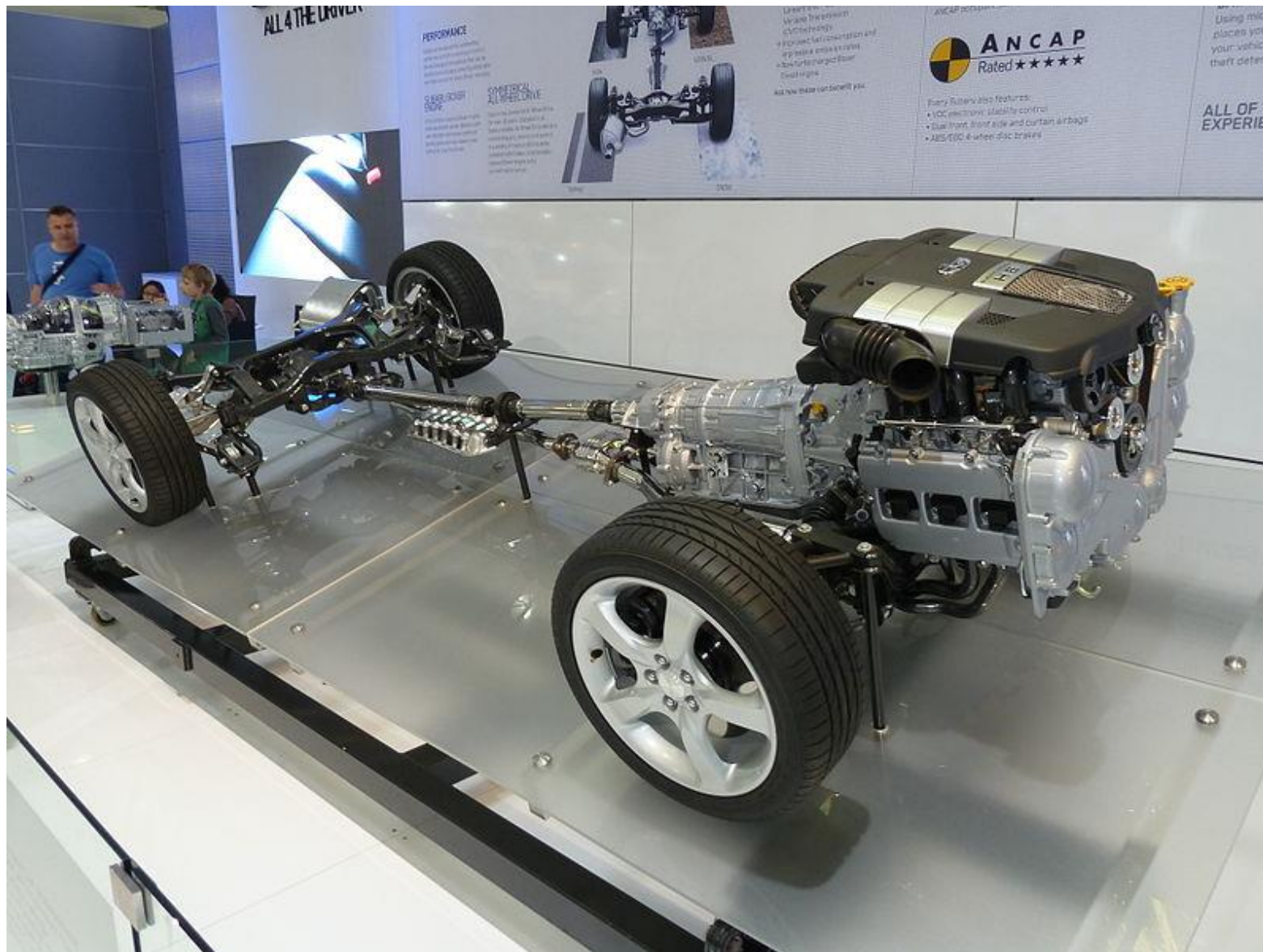
Vehicle Driveline

A brief Introduction

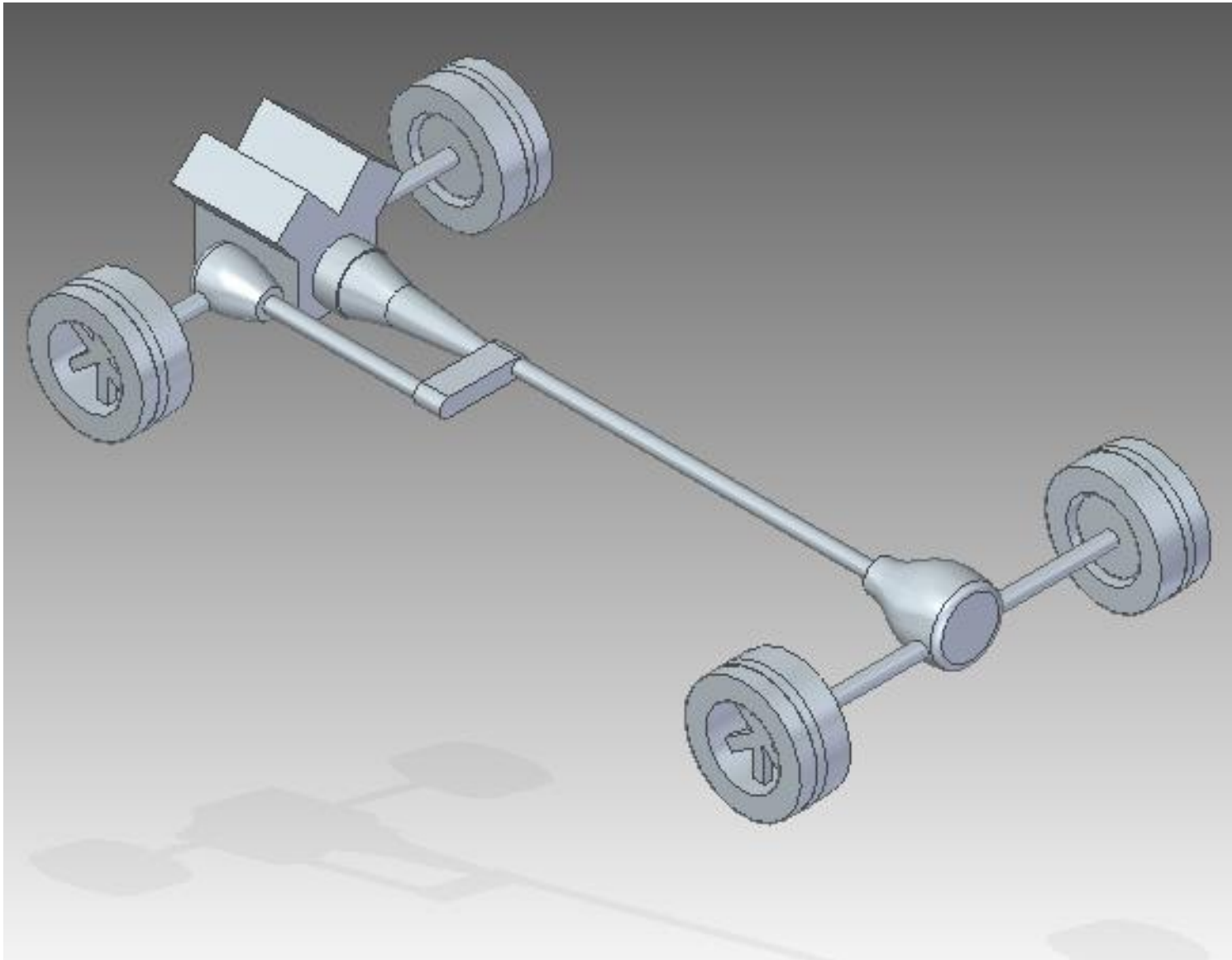
In a motor vehicles, the term vehicle driveline describes the main components that generate torque and deliver it to the wheels, which forms the interface.

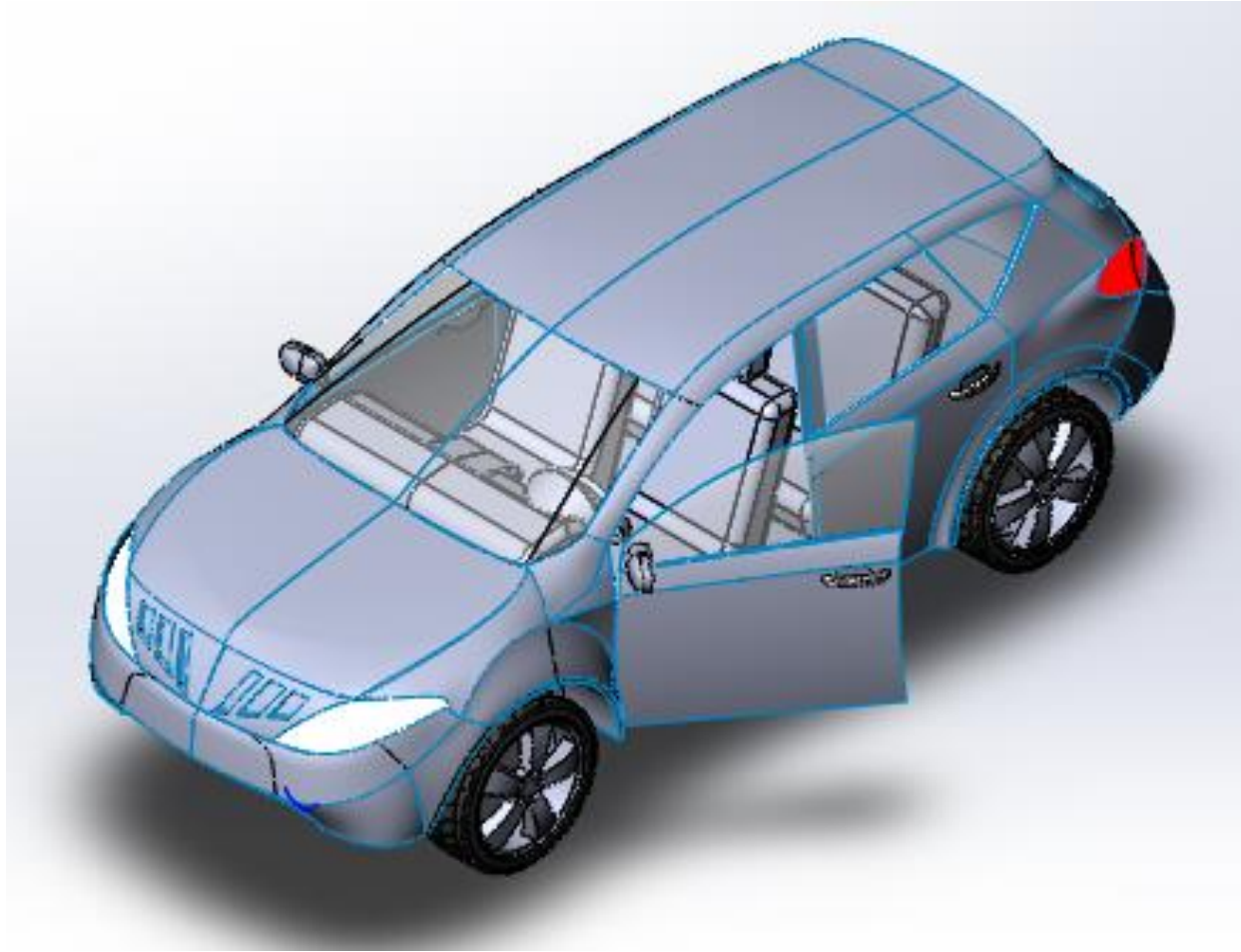
In layman language, this is the medium through which you connect the engine with the wheels.



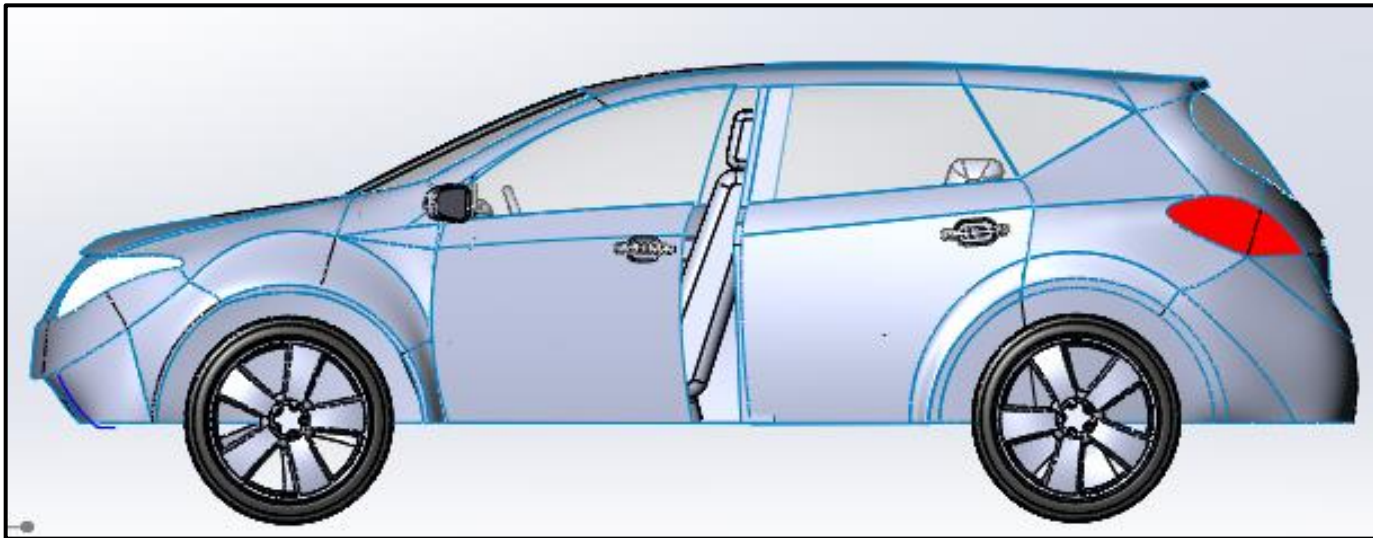


CAD MODEL OF DEIVELINE





Components of Drive Train



CAD MODEL OF CAR



Cadillac CTS Wagon

2010 GM 3.6L V-6 VVT DI (LLT)

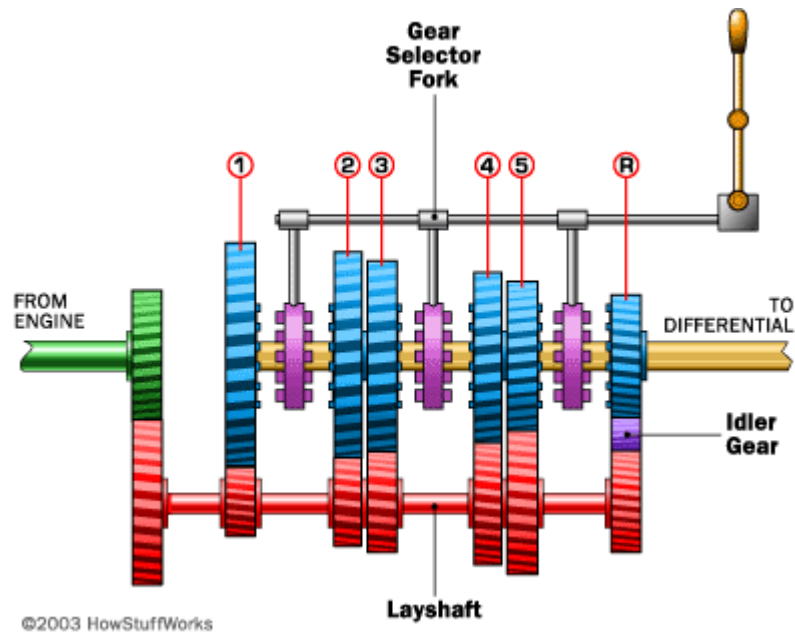
In vehicle we use Internal combustion engines. These convert the chemical energy present in fuel into mechanical energy.

At around 100 km/hr the passenger cars produce about 7000 rpm.



It transmits engine power to the gearbox, and allows transmission to be interrupted while a gear is selected to move off from a stationary position, or when gears are changed while the car is moving.

Most cars now a days use a friction clutch operated either by fluid (hydraulic).



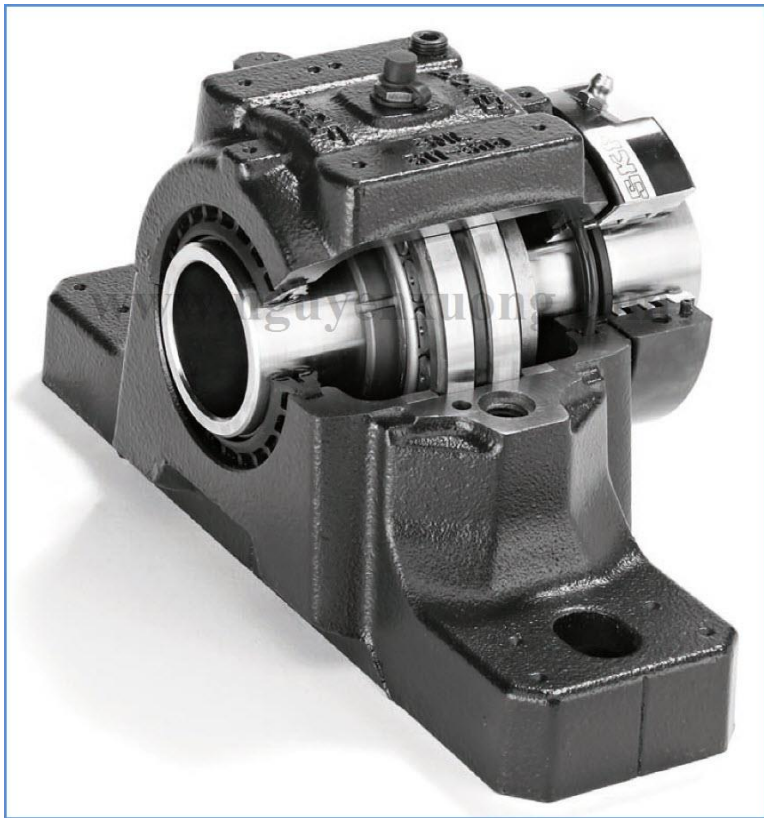
Engine torque is transmitted through the clutch to the transmission or transaxle. The transmission contains sets of gears that increase or decrease the torque, before it is transmitted to the rest of the drive train. The lower the gear ratio selected, the higher the torque transmitted.



It is used to transmit the torque from the gear box or transmission to the final drive.

Usually around 4 feet for cars to about 9 feet for trucks





The final drive provides a final gear reduction, to multiply the torque before applying it to the driving axles.

Inside the final drive, a differential gear set divides the torque to the axles, and allows for the difference in speed of each wheel when cornering.



It transmits the differential torque to driving wheels.
These along with the propeller shafts experience the highest twist



Material”

Driveline modeling

Model 1: Drive Shaft Flexibility

The clutch and the propeller shafts are assumed to be stiff, and the drive shaft is described as a damped torsional flexibility. The transmission and the final drive are assumed to multiply the torque with the conversion ratio, without losses.

$$M_w = M_d = k(\theta_f - \theta_w) + c(\dot{\theta}_f - \dot{\theta}_w)$$

$$M_w = M_d = k\left(\frac{\theta_m}{\tau_f \tau_g} - \theta_w\right) + c\left(\frac{\dot{\theta}_m}{\tau_f \tau_g} - \dot{\theta}_w\right)$$

$$(J_f + J_t \tau_f^2) \ddot{\theta}_m = M_c \tau_g^2 \tau_f^2 - b_t \dot{\theta}_m \tau_f^2 - b_f \dot{\theta}_m - k(\theta_m - \theta_w \tau_f \tau_g) - c(\dot{\theta}_m - \dot{\theta}_w \tau_f \tau_g)$$

This forms a state space form.

$$\dot{x} = Ax + BU + Hl$$

$$x_1 = \frac{\theta_m}{\tau_f \tau_g} - \theta_w \quad \begin{matrix} x_2 = \dot{\theta}_m \\ x_3 = \dot{\theta}_w \end{matrix}$$

Where

$$A = \begin{bmatrix} 0 & \frac{1}{\tau} & -1 \\ \frac{-k}{\tau J_1} & \frac{-(B_1 + c/\tau^2)}{J_1} & \frac{c}{\tau J_1} \\ \frac{k}{J_2} & \frac{c}{\tau J_2} & \frac{-(c + b_2)}{J_2} \end{bmatrix} \quad B = \frac{1}{J_1} \quad H = \begin{bmatrix} 0 \\ 0 \\ -1 \\ J_2 \end{bmatrix}$$

$$l = r_w m(c_{r1} + g \sin \alpha)$$

$$b_1 = \frac{b_t}{\tau_g^2} + \frac{b_f}{\tau_g^2 \tau_f^2}$$

$$J_2 = J_w + m r_w^2$$

$$b_2 = b_w + m c_{r2} r_w^2$$

$$J_1 = J_m + \frac{J_t}{\tau_g^2} + \frac{J_f}{\tau_f \tau_g}$$

$$\tau = \tau_t = \tau_f \tau_g$$

Model 2: Flexible Clutch and Drive Shafts

A model with a linear clutch and one torsional flexibility (the drive shaft) is derived by repeating the procedure for Model 1 with the difference that the model for the clutch is a flexibility with stiffness k_c and internal damping c_c .

$$(J_t + J_f/i_f^2)\ddot{\theta}_t = i_t \left(k_c(\theta_m - \theta_t i_t) + c_c(\dot{\theta}_m - \dot{\theta}_t i_t) \right) - (b_t + b_f/i_f^2)\dot{\theta}_t - M_d/i_f$$

Model 3: Flexible Propeller Shaft

A model with a flexible propeller and drive shaft, is derived by repeating the procedure for Model 1 with the difference that the model for the propeller shaft is a flexibility with stiffness k_p and internal damping c_p .

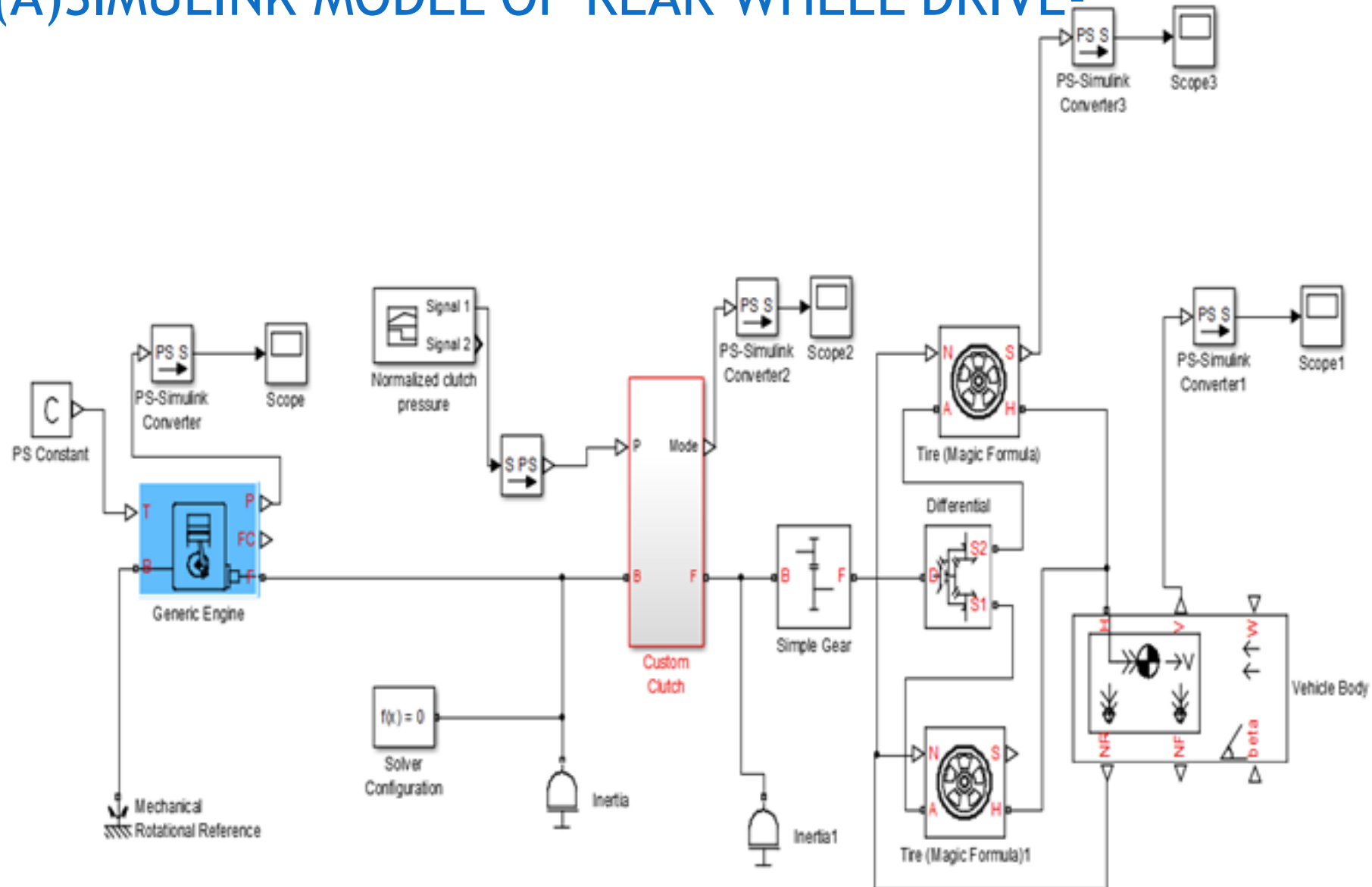
$$(J_w + mr_w^2)\ddot{\theta}_w = k_d(\theta_p/i_f - \theta_w) + c_d(\dot{\theta}_p/i_f - \dot{\theta}_w) \\ - b_w\dot{\theta}_w - \frac{1}{2}c_w A_a \rho_a r_w^3 \dot{\theta}_w^2 - mc_{r2}r_w^2\dot{\theta}_w - r_w m(c_{r1} + g\sin(\alpha))$$

PERFORMANCE ANALYSIS OF DRIVELINE SYSTEM USING SIMDRIVELINE

Parameters and Assumptions

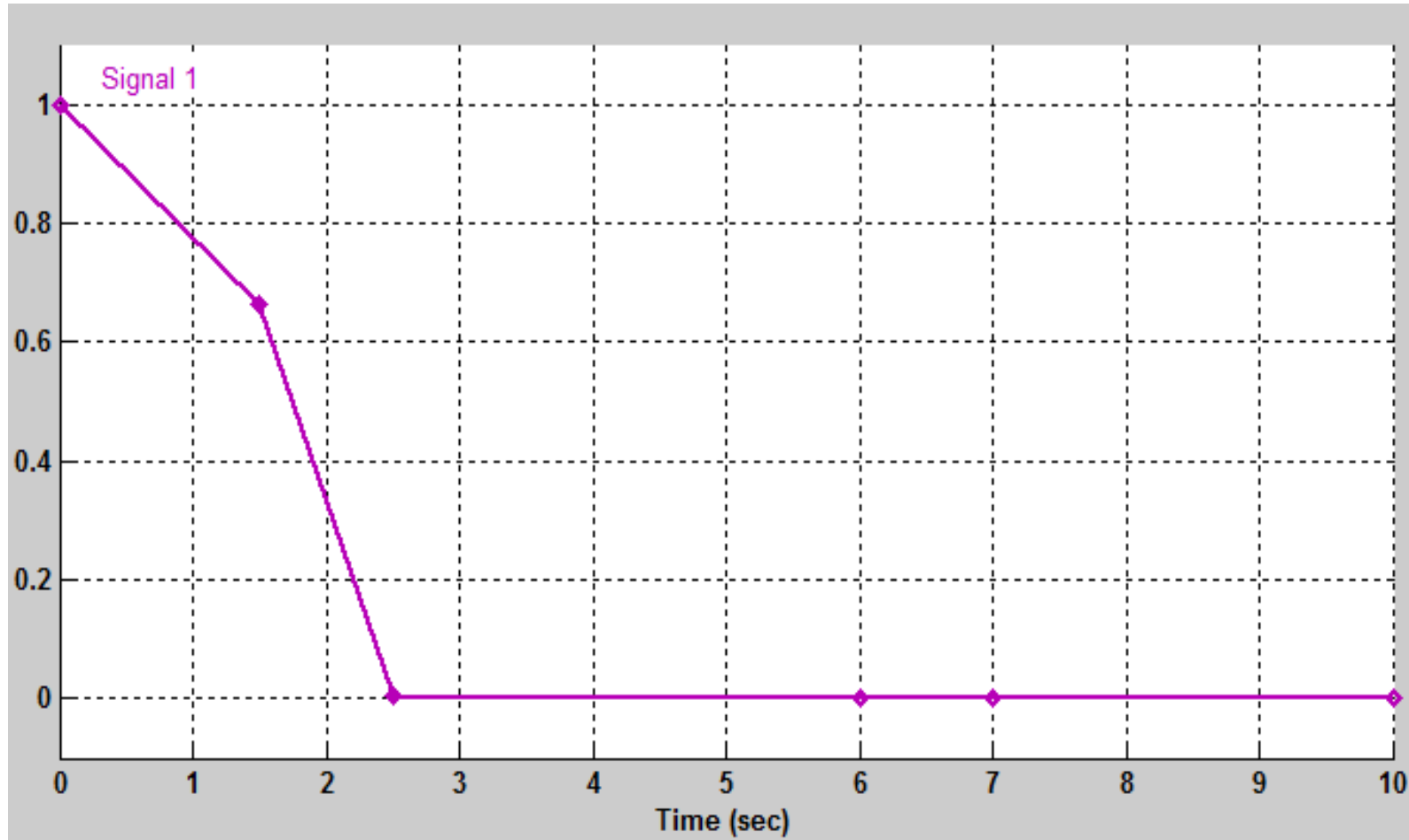
- ▶ Engine type: spark engine
- ▶ Maximum power : 45000W
- ▶ Speed at maximum power : 3800rpm
- ▶ Maximum speed : 6000rpm
- ▶ Stall speed : 350 rpm
- ▶ Engine inertia : 0.2 kg-m^2
- ▶ Fuel consumption per revolution: 25mg/rev
- ▶ Inertia of shaft 1 : 0.2 kg-m^2
- ▶ Inertia of shaft 2 : 1 kg-m^2
- ▶ Single Gear, No meshing and viscous losses
- ▶ Single Custom clutch
- ▶ Throttle 100 % open for the engine
- ▶ No flexibility assumed for any component

(A)SIMULINK MODEL OF REAR WHEEL DRIVE-



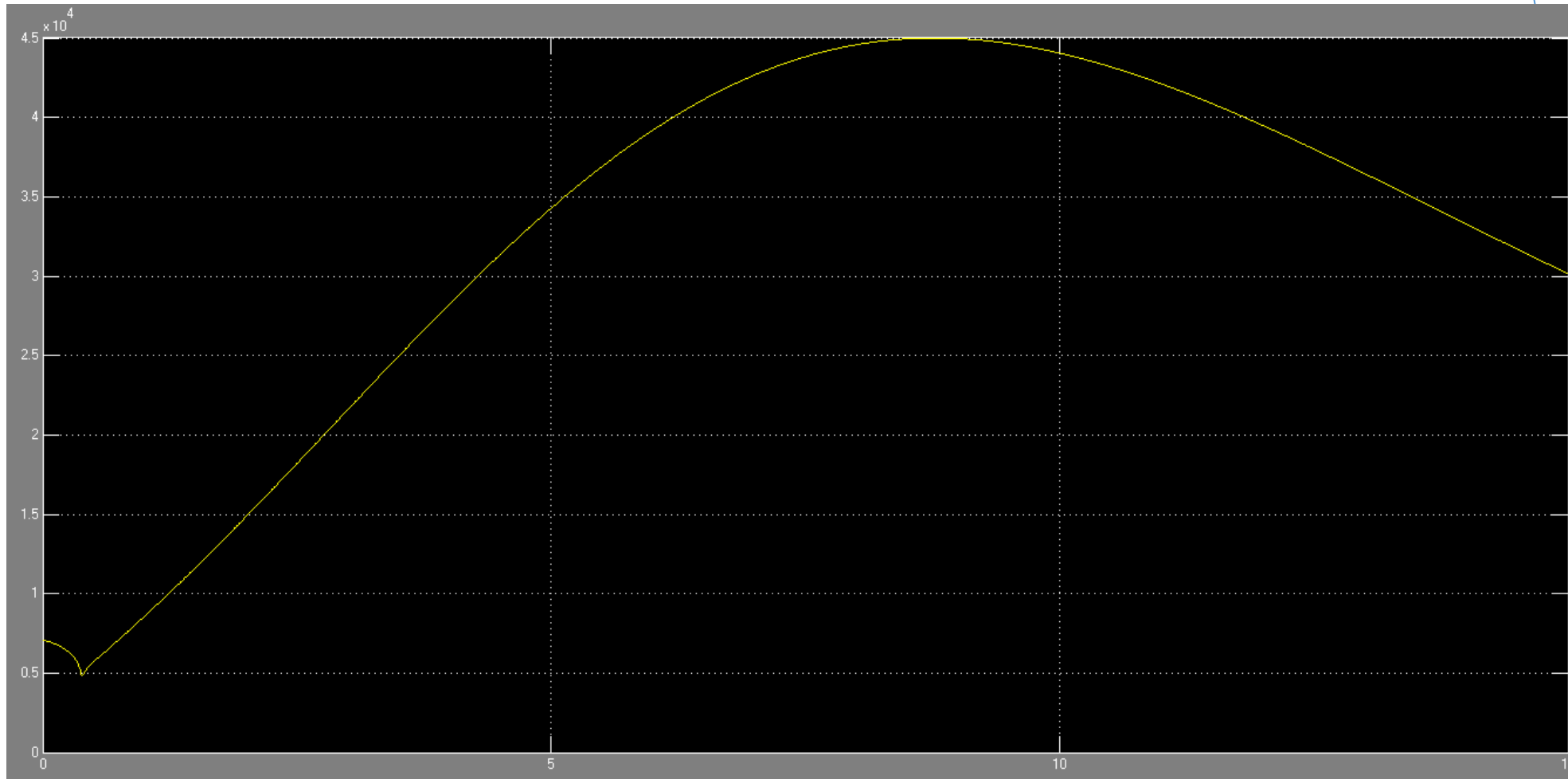
1.INPUT OF CLUTCH

Normalized pressure vs time(sec)



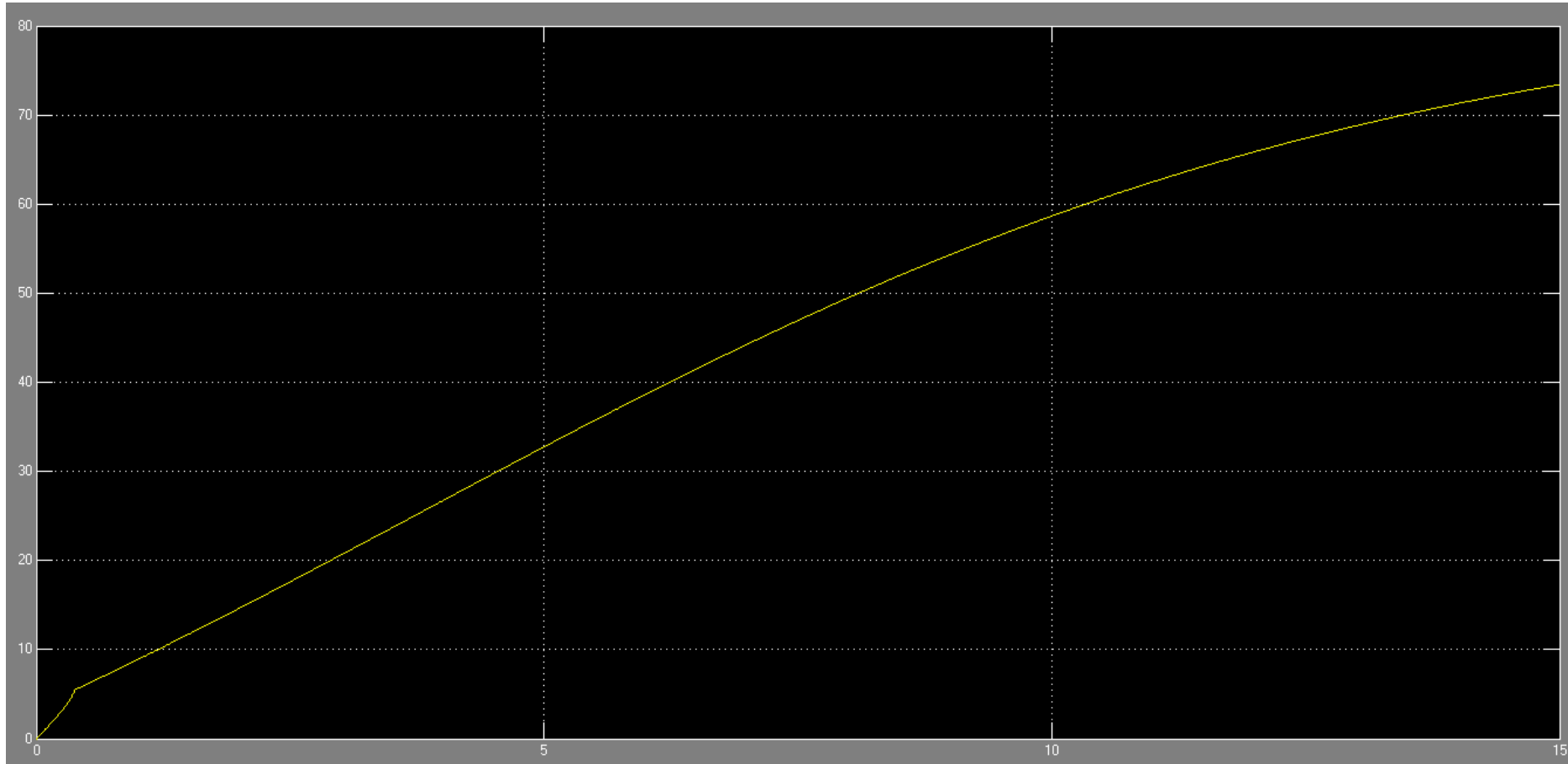
2. POWER OUTPUT OF ENGINE

Power output (in Watt) vs time(in sec)



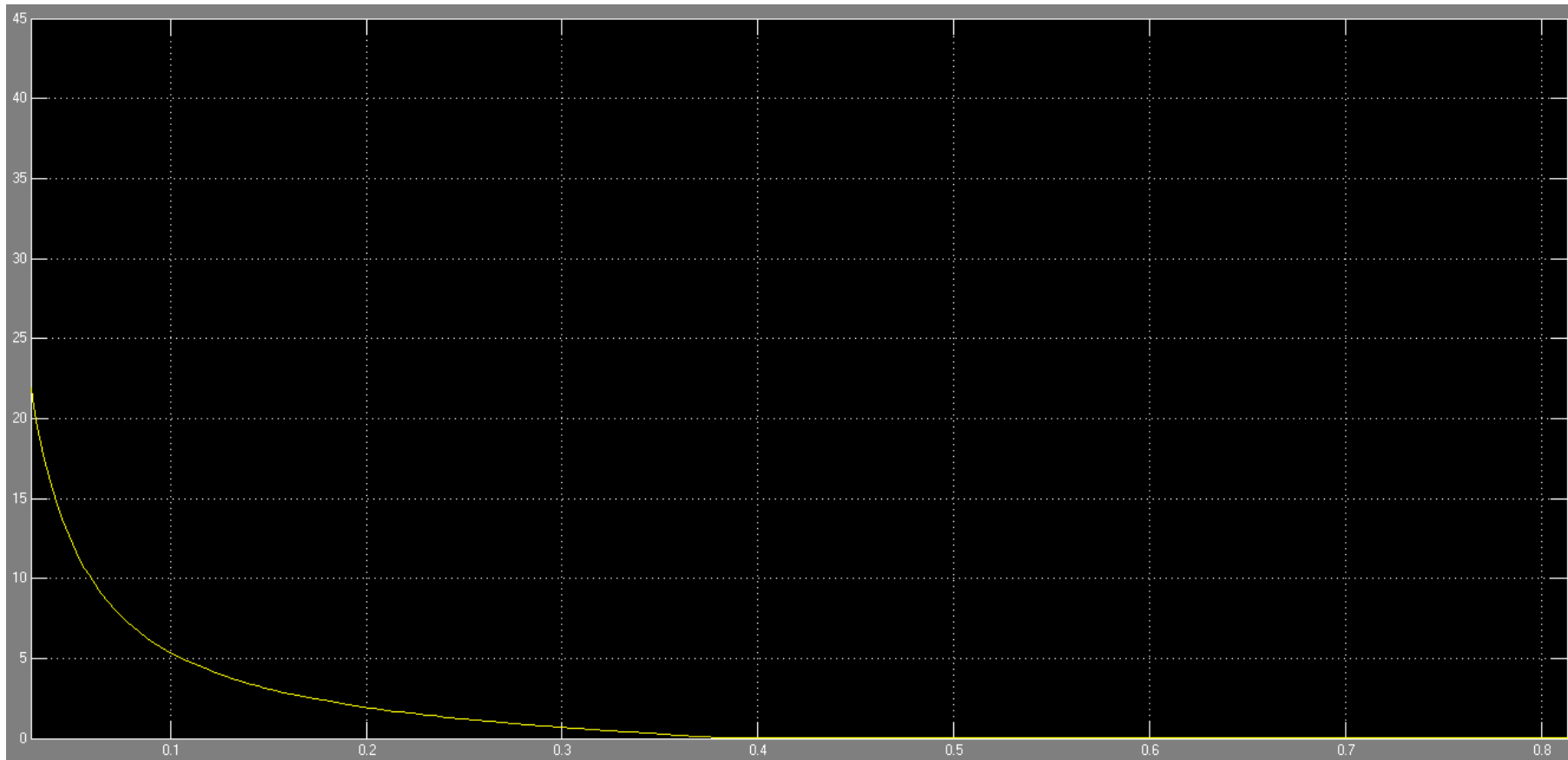
3. VELOCITY OF VEHICLE

Velocity (in km/hr) vs time (in sec)



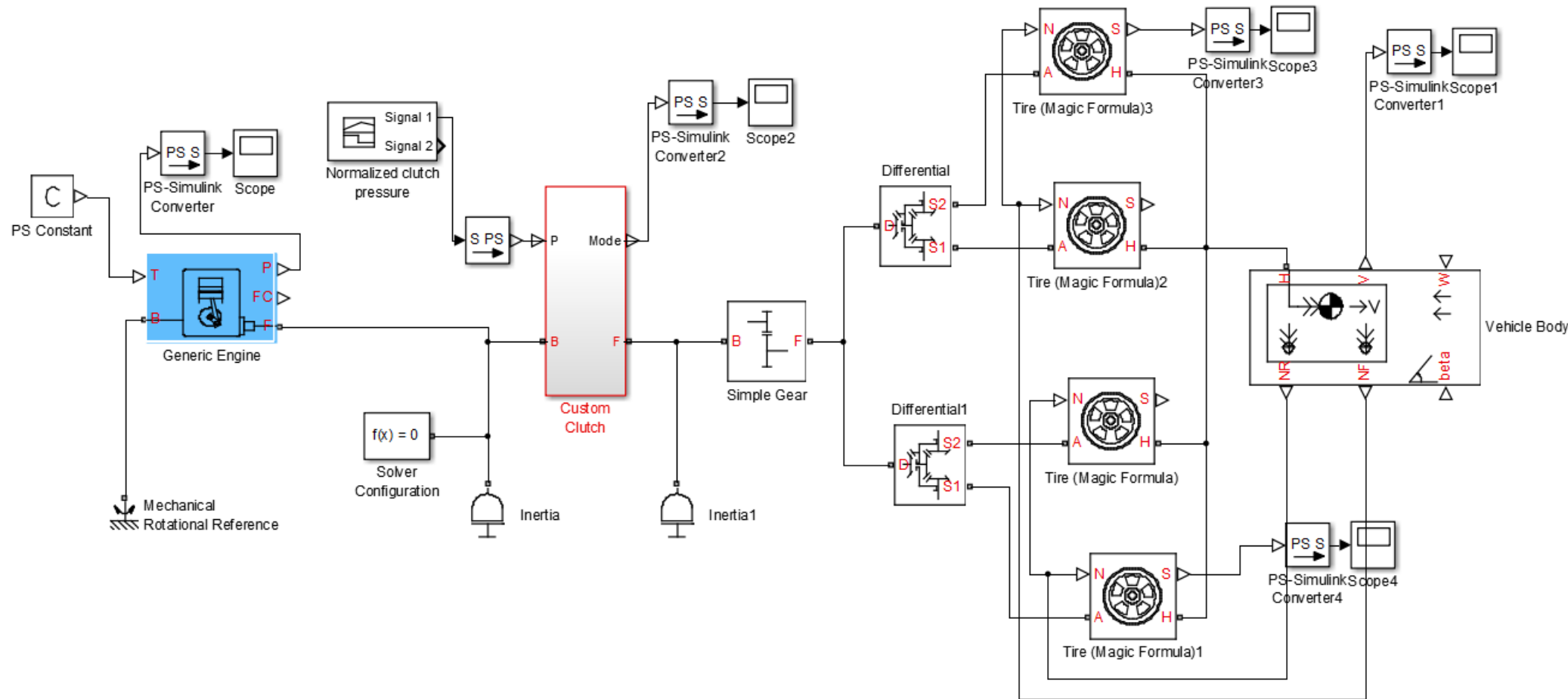
4. SLIP OF THE WHEEL

Tire slip vs time(in sec)

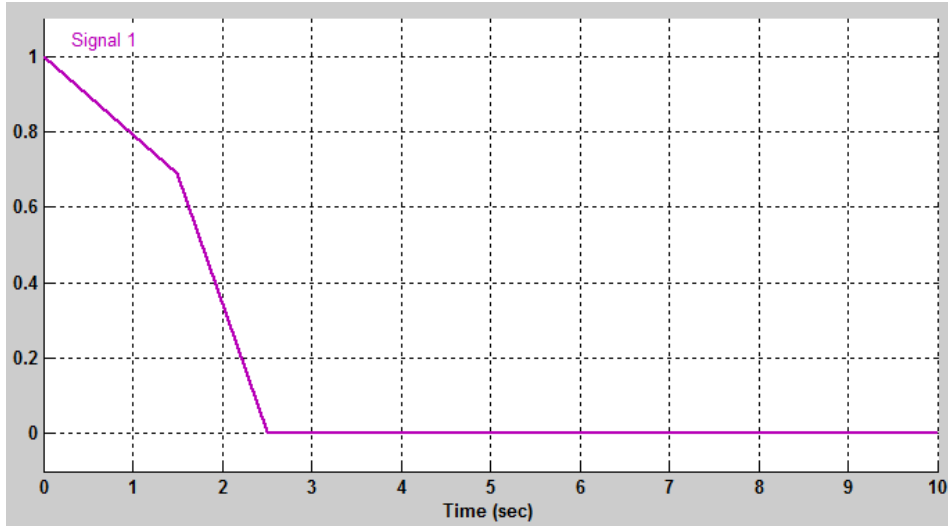


(B) SIMULINK MODEL OF FOUR WHEEL DRIVE

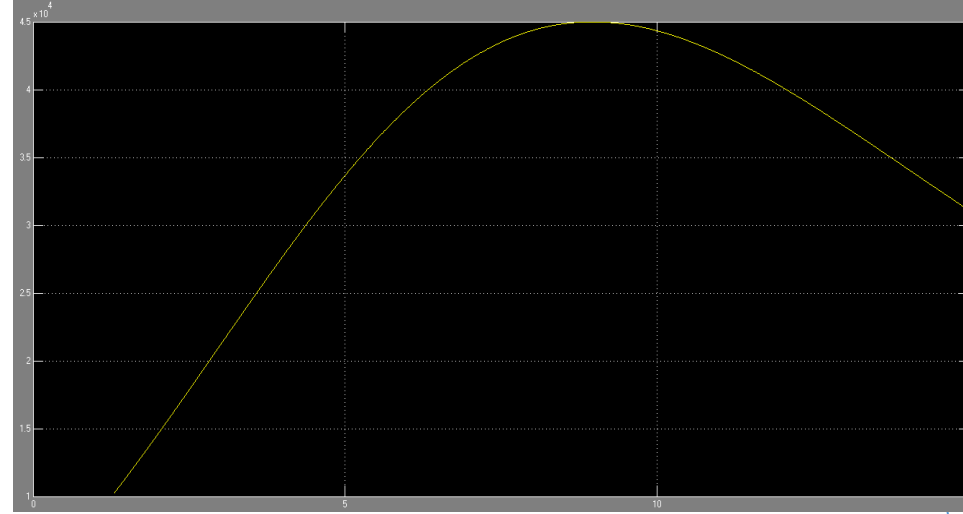
(Using same parameters and assumptions)



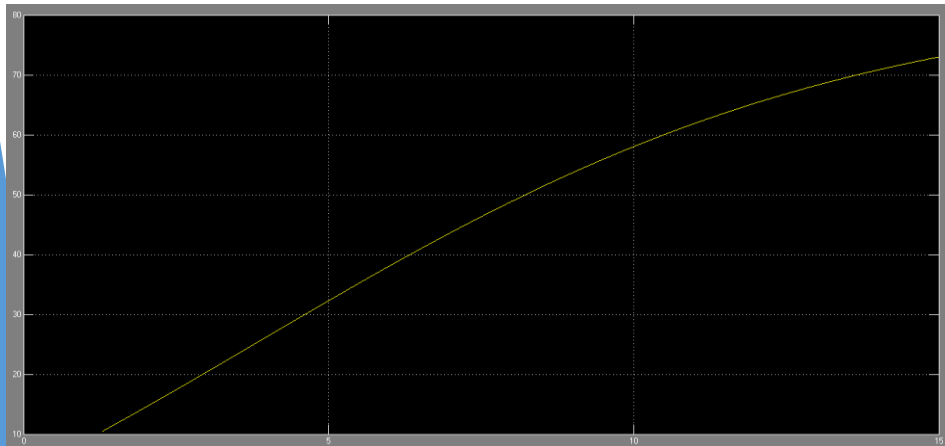
Clutch pressure vs time(in sec)[input]



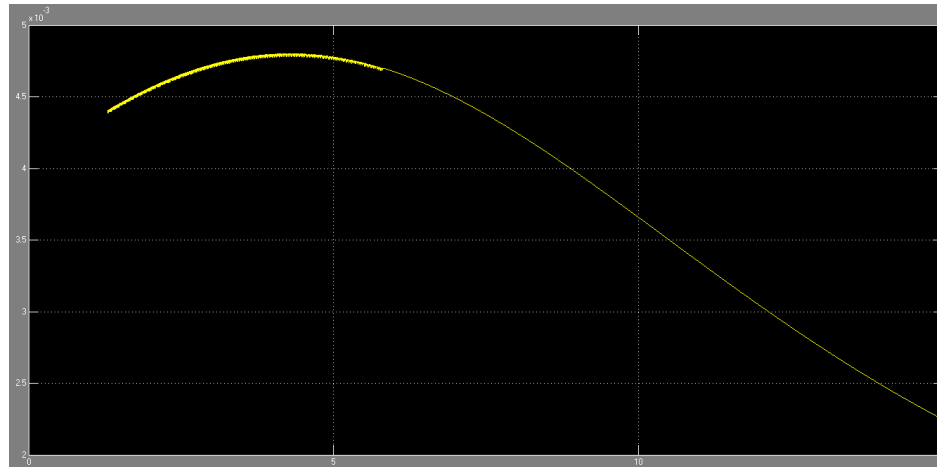
Engine power(Watt) vs Time(in sec)



Vehicle velocity(km/hr) vs time(in sec)



Tire slip(10^{-3}) vs time(in sec)



Four wheel and Two wheel drive

Four wheel drive (4WD) is a form of all-wheel drive powertrain capable of providing power to all wheels of a four wheeled vehicle simultaneously. The car has a way to send engine power to both the front and rear tires. Four Wheel drive means all four wheels are motorized, and that gives better traction in extreme conditions.

Two-wheel drive (FWD) is a form of engine and transmission layout used in motor vehicles, where the engine drives the front wheels only. The direct connection between engine and transaxle reduces the mass and mechanical inertia of the drivetrain.

COMPARISON OF TWO WHEEL AND FOUR WHEEL DRIVE USING CARSIM

Four wheel
Front wheel



Parameters for four wheel drive

Vehicle Body

Sprung mass: Rigid Sprung Mass
A-Class, Hatchback Sprung Mass

Aerodynamics
A-Class, Hatchback Aero

Animator Data

Vehicle animator data: Vehicle Shape
A-Class, Subcompact

Systems

Powertrain: 4-wheel drive
150 kW, 6-spd., 4.1 Ratio

Brake system: 4-wheel system
B-Class, Hatchback w/ ABS

Steering system: 4-wheel steer
A-Class, Hatchback: Manual, R&P

Misc. keywords and values:

☐ 3x1 image scale



Front

Suspension type: Independent

Front kinematics: Independent
A-Class, Hatchback - Front Suspension

Front compliance: Independent
A-Class, Hatchback - Front Comp.

Right-front tire: Tire
175/65 R14

Left-front tire: Tire
175/65 R14

Rear

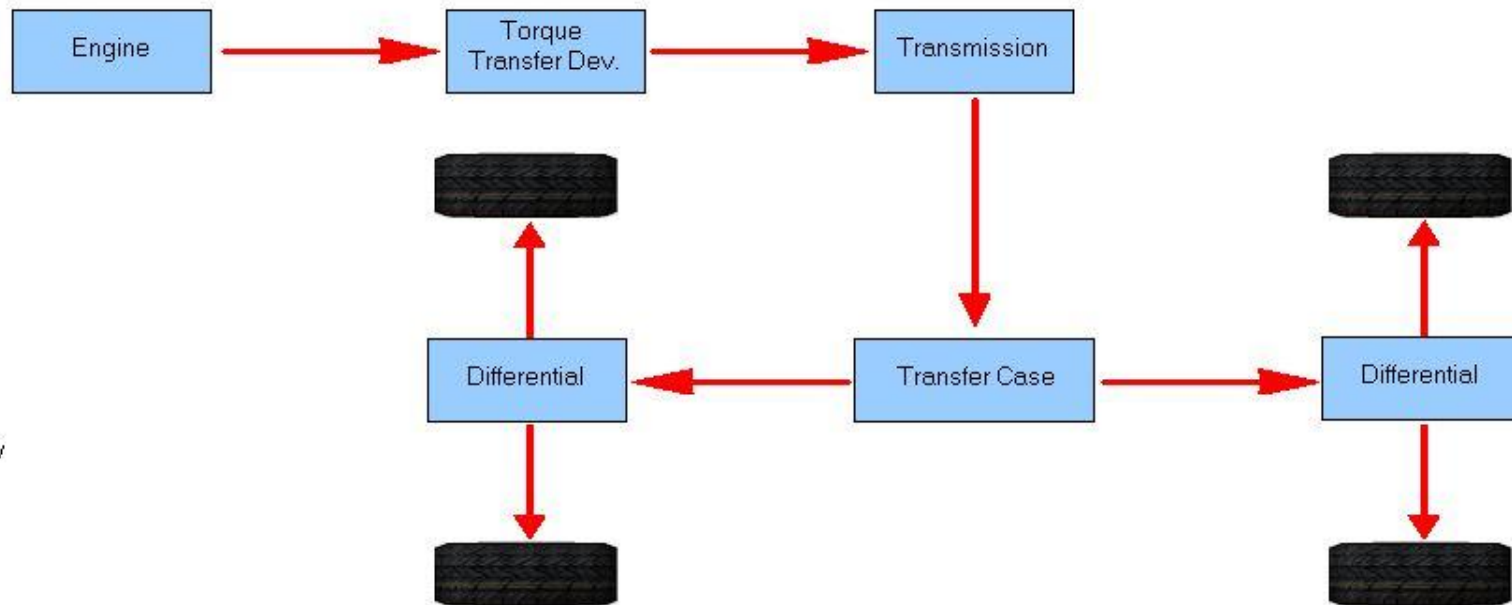
Suspension type: Independent

Rear kinematics: Independent
A-Class, Hatchback - Rear Suspension

Rear compliance: Independent
A-Class, Hatchback - Rear Comp.

Right-rear tire: Tire
175/65 R14

Left-rear tire: Tire
175/65 R14



Internal engine model
150 kW Engine

Internal torque converter model
Torque Converter for 150 kW Engine

Internal transmission model (up to 18 gears)
6-Speed Transmission

Internal transfer case
Full Time Viscous 50/50

Internal front differential
Open - Gear Ratio 4.1

Internal rear differential
Open - Gear Ratio 4.1

Drive shaft roll effect

Cause positive roll

The drive shaft applies a roll moment between the engine mass and differential. Most drivelines cause positive roll of the engine mass during acceleration. (When viewed from the front of the vehicle, the drive shaft rotation is clockwise.)

☐ Front drive torque pitch effect

☐ Rear drive torque pitch effect

Parameters for front wheel drive

Vehicle Body

Sprung mass: Rigid Sprung Mass

A-Class, Hatchback Sprung Mass

Aerodynamics

A-Class, Hatchback Aero

Animator Data

Vehicle animator data: Vehicle Shape

A-Class, Subcompact

Systems

Powertrain: Front-wheel drive

150 kW, 6-spd., 4.1 Ratio

Brake system: 4-wheel system

B-Class, Hatchback w/ ABS

Steering system: 4-wheel steer

A-Class, Hatchback: Manual, R&P

Misc. keywords and values

☐ 3x1 image scale



Front

Suspension type: Independent

Front kinematics: Independent

A-Class, Hatchback - Front Suspension

Front compliance: Independent

A-Class, Hatchback - Front Comp.

Right-front tire: Tire

175/65 R14

Left-front tire: Tire

175/65 R14

Rear

Suspension type: Independent

Rear kinematics: Independent

A-Class, Hatchback - Rear Suspension

Rear compliance: Independent

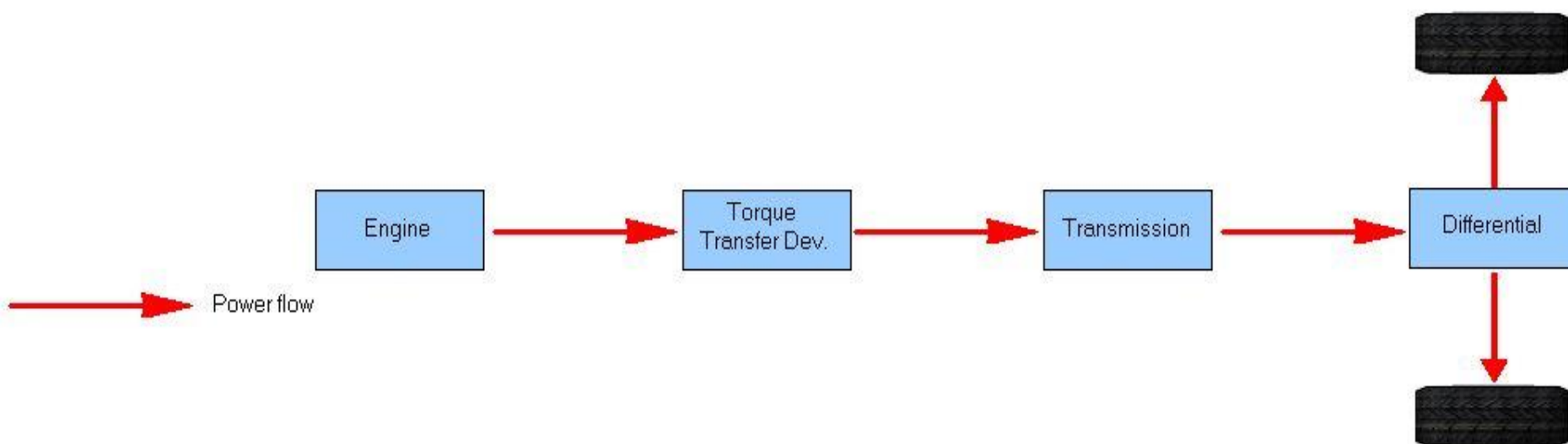
A-Class, Hatchback - Rear Comp.

Right-rear tire: Tire

175/65 R14

Left-rear tire: Tire

175/65 R14



Internal engine model
150 kW Engine

Internal torque converter model
Torque Converter for 150 kW Engine

Internal transmission model (up to 18 gears)
6-Speed Transmission

Internal differential
Open - Gear Ratio 4.1

Driveline Torsional Flexibility

This flexibility involves all driveline parts (incl. transmission, differentials, and all drive wheels, but not engine).

Drive shaft roll effect

Cause positive roll

The drive shaft applies a roll moment between the engine mass and differential. Most drivelines cause positive roll of the engine mass during acceleration. (When viewed from the front of the vehicle, the drive shaft rotation is clockwise.)

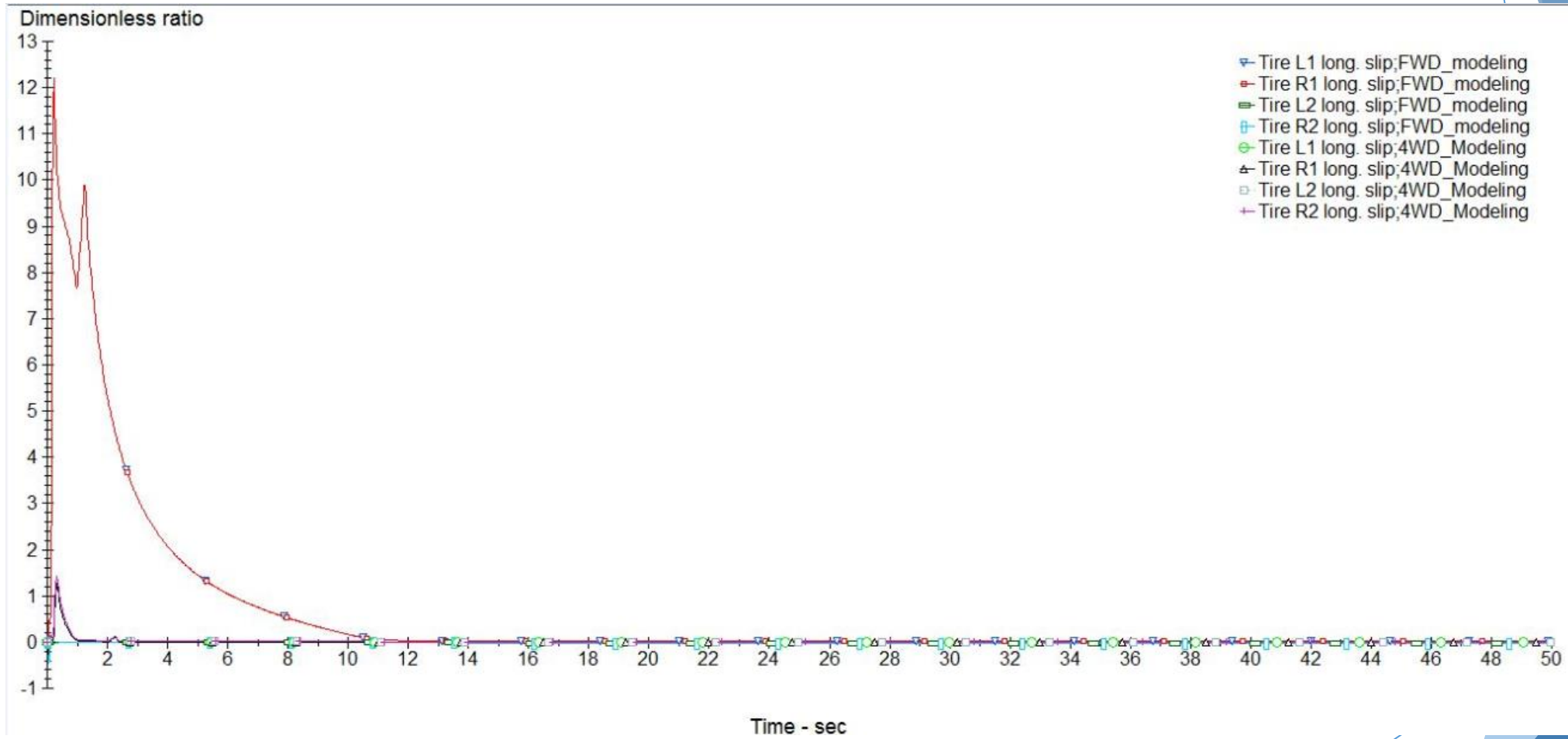
☐ Drive torque pitch effect to engine body

Check this box if the engine body has a pitch effect from

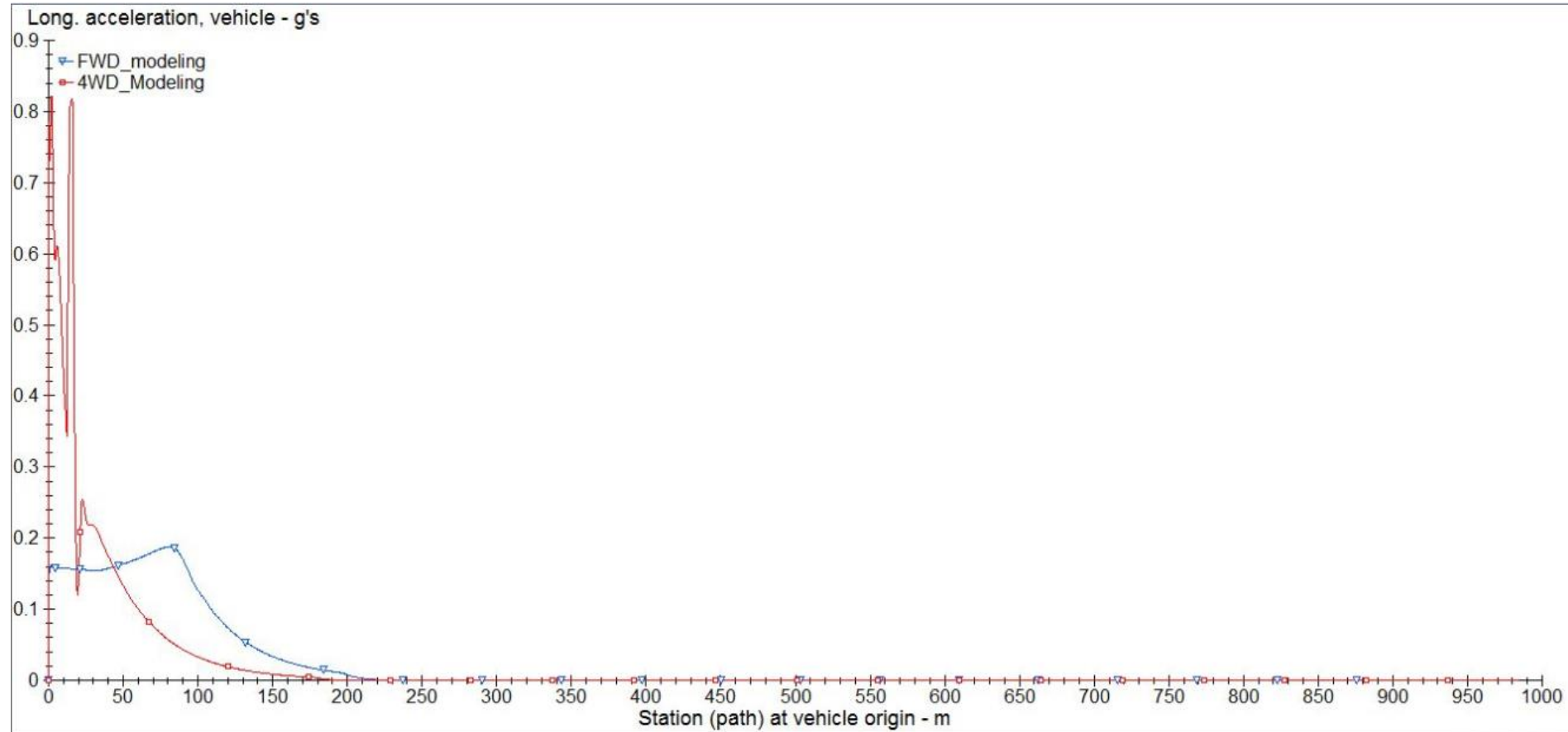
Driver Controls

Driver Controls	Start and Stop Conditions									
Target speed from path preview ▼	Stop run at specified time ▼									
Normal Driving ▼	<table><thead><tr><th></th><th>Time (sec)</th><th>Path station (m)</th></tr></thead><tbody><tr><td>Start:</td><td>0</td><td>0</td></tr><tr><td>Stop:</td><td>50</td><td>Road forward ▼</td></tr></tbody></table>		Time (sec)	Path station (m)	Start:	0	0	Stop:	50	Road forward ▼
	Time (sec)	Path station (m)								
Start:	0	0								
Stop:	50	Road forward ▼								
Braking: Brake control ▼	<input type="checkbox"/> Specify initialization details?									
No Open-Loop Braking Pressure ▼										
Shifting control: Closed-loop shift control ▼										
AT All Gears ▼										
Steering: ▼										
	Additional Data									

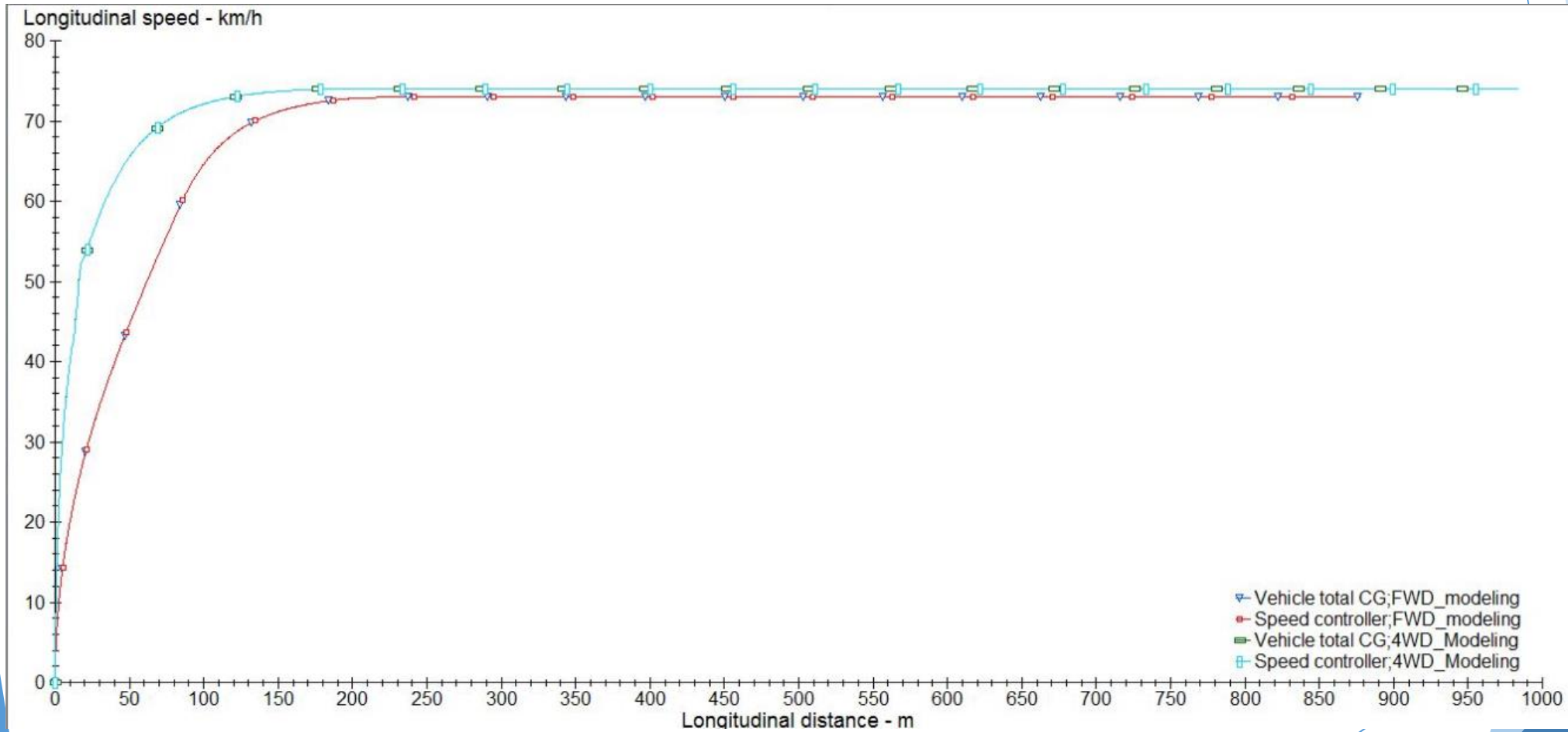
1.Slip vs time-



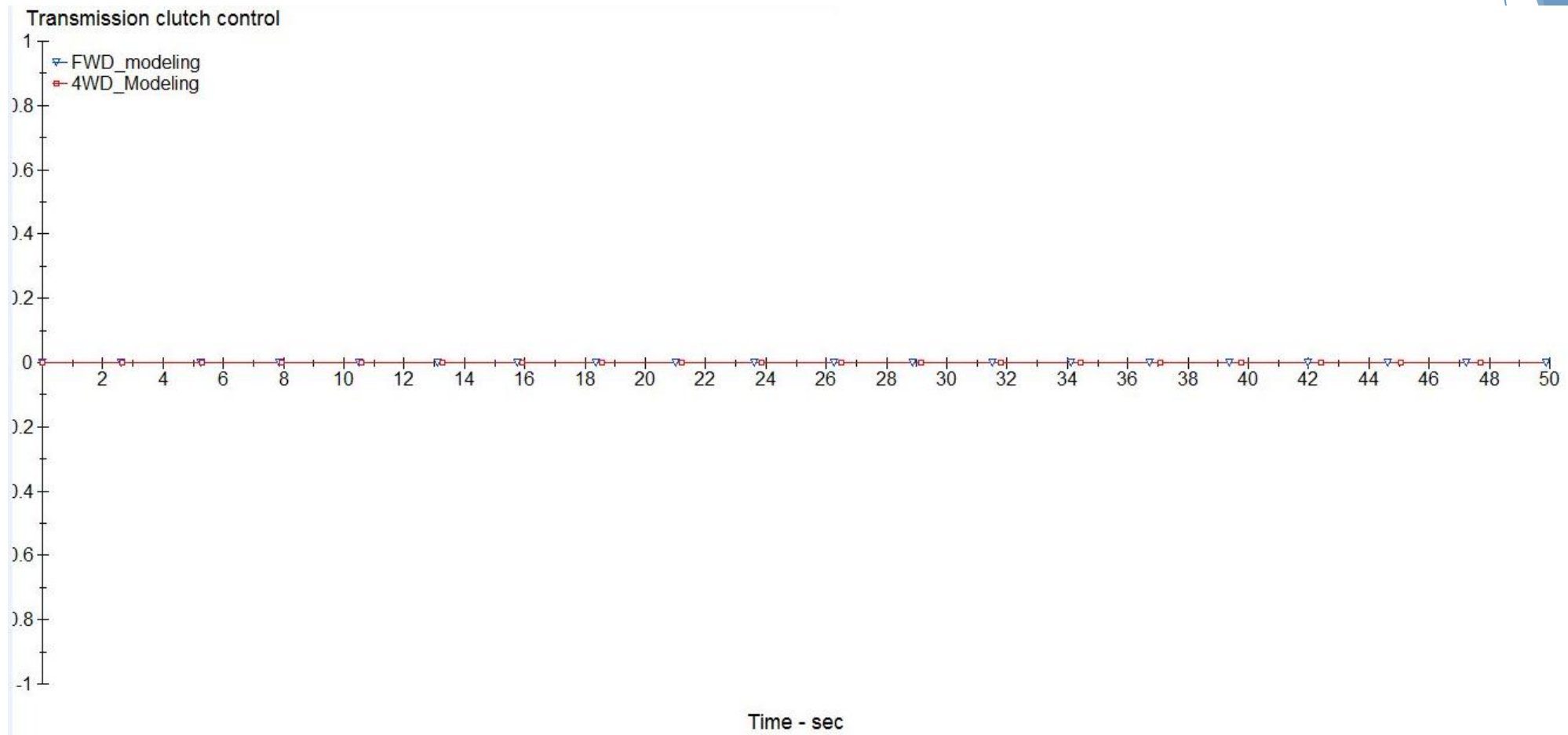
2. Acceleration vs distance-



3. Longitudinal velocity vs distance-



4. Transmission clutch control vs time-



REFERENCES-

<http://in.mathworks.com/help/physmod/sdl/ug/specialized-and-customized-driveline-components.html>
<http://in.mathworks.com/help/physmod/sdl/examples.html>
<http://in.mathworks.com/help/physmod/sdl/ug/complete-car-model-and-simulation.html>
<http://in.mathworks.com/help/physmod/sdl/examples/simple-transmission.html>
<http://in.mathworks.com/help/physmod/sdl/ref/genericengine.html>
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<http://citeseerx.ist.psu.edu/viewdoc/download?doi=10.1.1.39.9750&rep=rep1&type=pdf>
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THANK YOU